



Istituto Nazionale di Fisica Nucleare  
SEZIONE DI FERRARA

The background of the slide is a light-colored technical drawing or architectural plan. Overlaid on this drawing are several drafting tools: a yellow pencil, a pair of compasses, and a set square. A bright orange beam of light enters from the left side, illuminating the tools and the drawing.

# dRICH

## Mechanics



- INFN-FE Mechanical Engineering Group
- dRICH Concept Design

# Mechanical Engineering Group (INFN-Ferrara Departement & UNIFE)

The ME Group works in close collaboration with researchers, from the early concept of a detector/device to its construction.

The group includes a design office and a mechanical workshop facility with state-of-the-art equipment.

The design office performs mechanical design, advanced calculations, analyses and numerical simulations using computer-aided design (CAD) and computer-aided engineering (CAE) software.

The mechanical workshop core competencies are conventional and high-precision computer numerical control (CNC) machining, sheet metal forming, welding engineering, micrometric 3D metrology and additive manufacturing.

## Team:

Federico Evangelisti, Michele Melchiorri: mechanical design

Stefano Squerzanti: mechanical design, CMM, AM

Michele Cavallina: mechanical workshop

Alessandro Saputi: head of ME Group

# Mechanical Engineering Group: internal workshop facilities

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AM Lab



Design Office



Mechanical workshop



Store



Metrology Lab



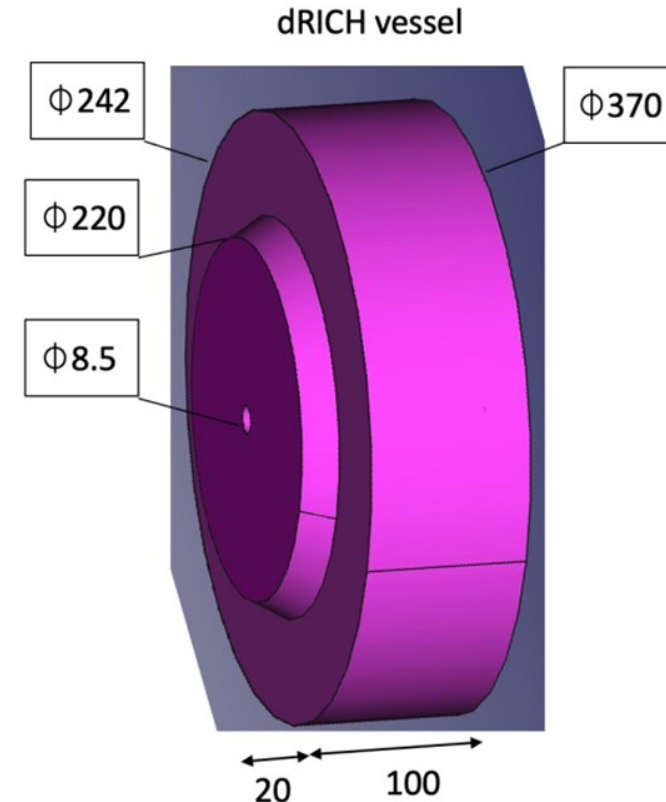
Welding Lab & Carpenter shop



Carpentry workshop

# dRICH: (preliminary) gas enclosure requirements

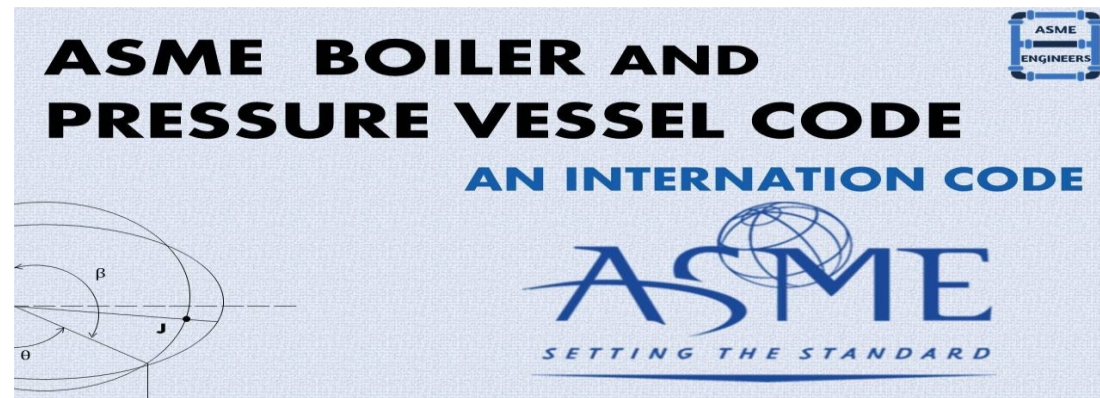
- The major functions of the gas enclosure are to provide containment for the dRICH gas radiator and to act as the stable frame for the optical components (the mirrors and aerogel).
- It must be light tight.
- To ensure the stability of the structure under the influence of the magnetic field.
- The enclosure must withstand a differential pressure ( $\pm 3$  mbar??) without compromising the mirror alignment.
- The minimum amount of material must be placed within the EPIC experiment acceptance limits.
- Envelope overall sizes:  $\Phi 3670$  mm x 1130 mm
- Operating pressure up to 2.5 atm abs ( $\sim 1.5$  atm – gauge pressure)
- Operating temperature of 22 °C
- Gas mixture: C2F6 or Argon



# dRICH: as a pressure vessel

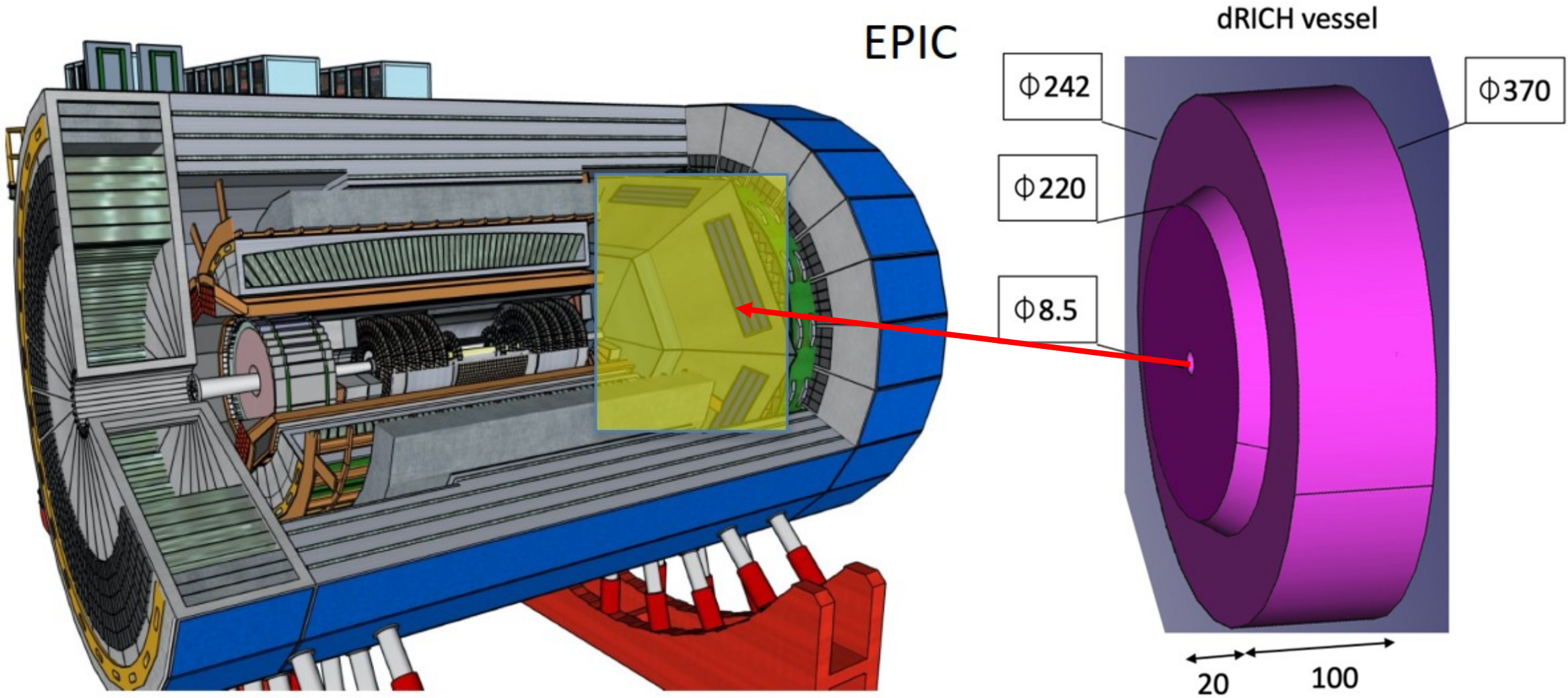
Operating pressure of 2.5 atm abs (~ 1.5 atm)

Design pressures exceeding 15 psig (1 atm) are generally ASME labelled and considered ASME pressure vessels

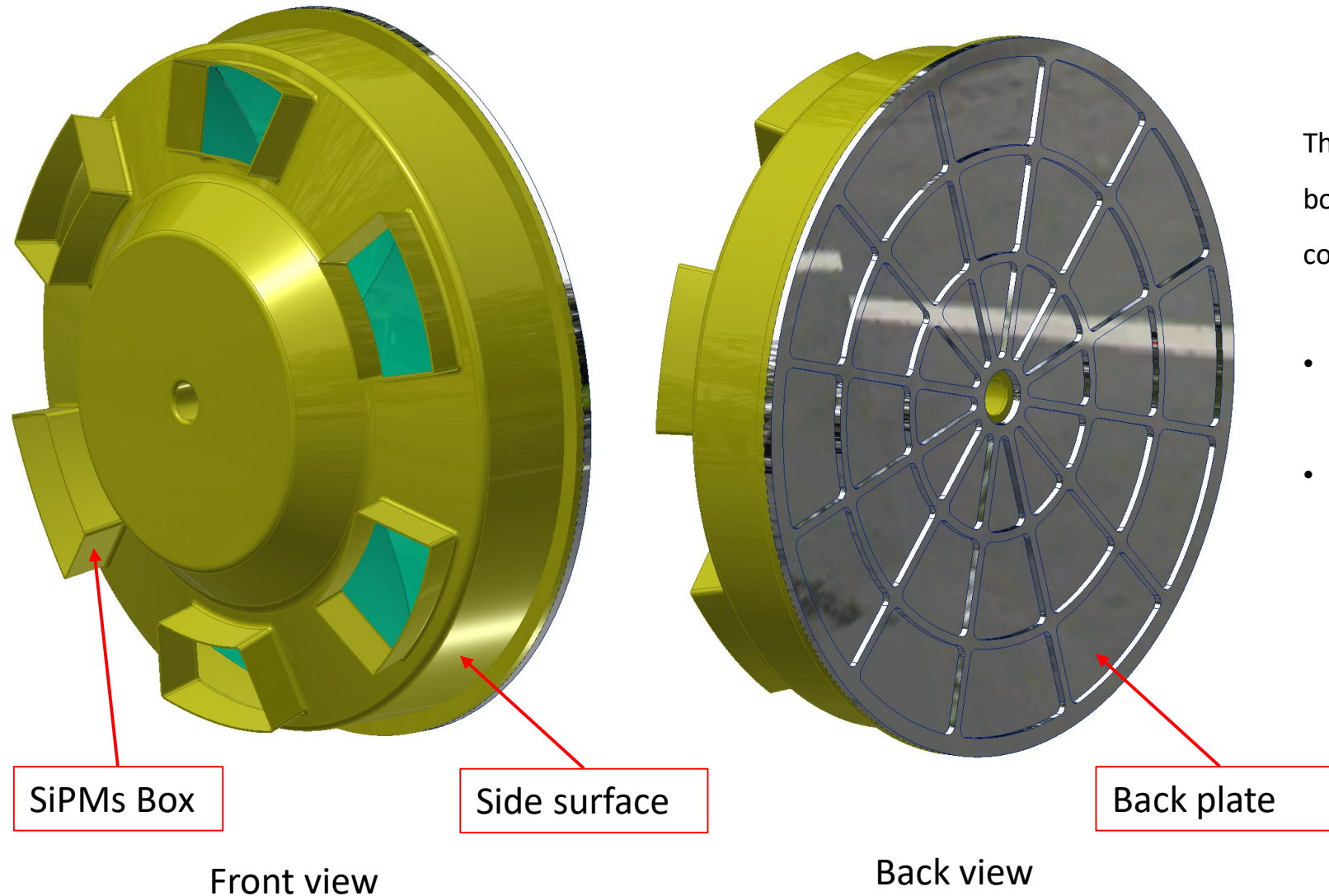


# dRICH: concept design

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# dRICH: concept design



The gas enclosure is essentially a cylindrical box with six boxes that host SiPMs and cooling system.

- The upstream face is closed by a plate which maintains the gas and light seal.
- The boxes are separated from gas volume by quartz windows (that allow Cherenkov photons to pass through to the SiPMs mounted behind).



# dRICH: concept design

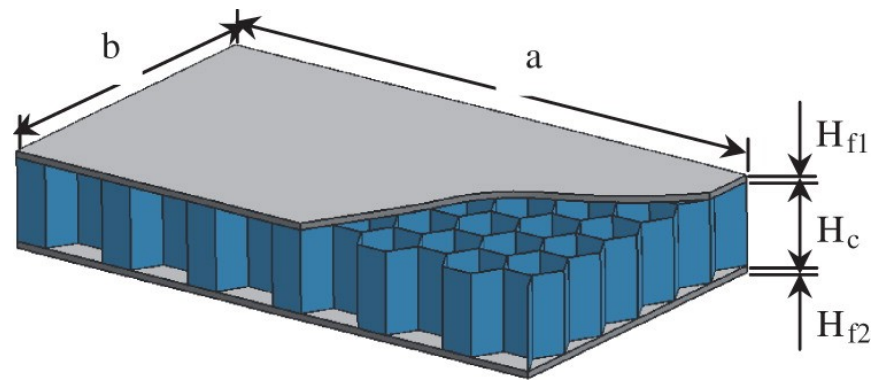
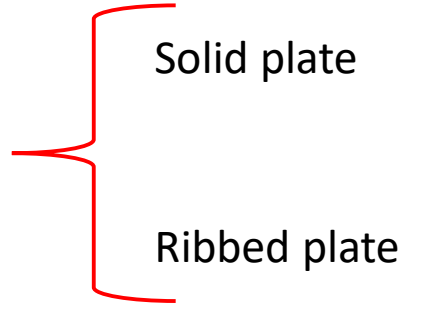
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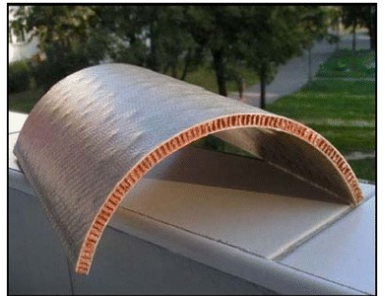
Metal



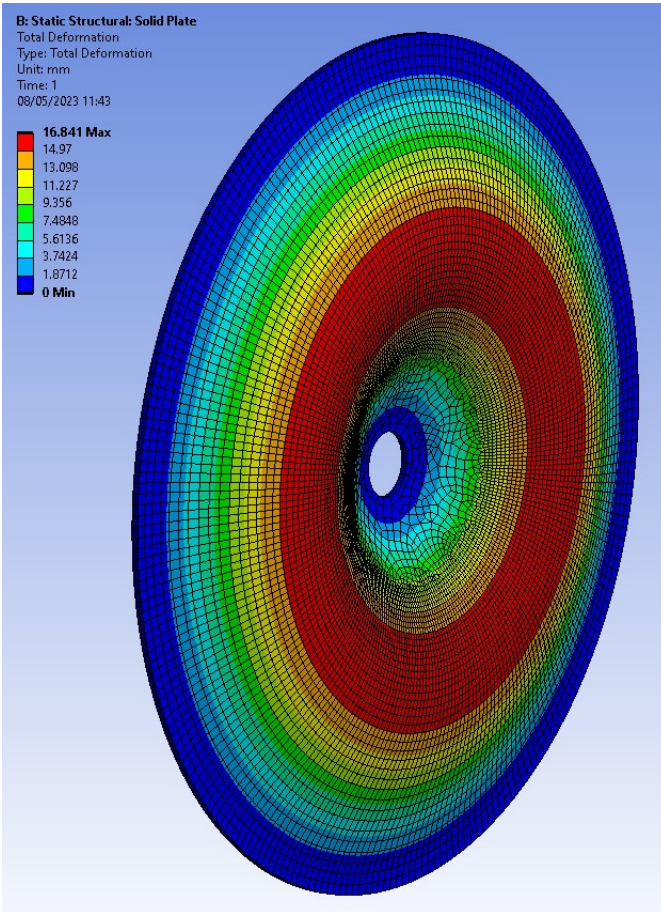
CF laminate



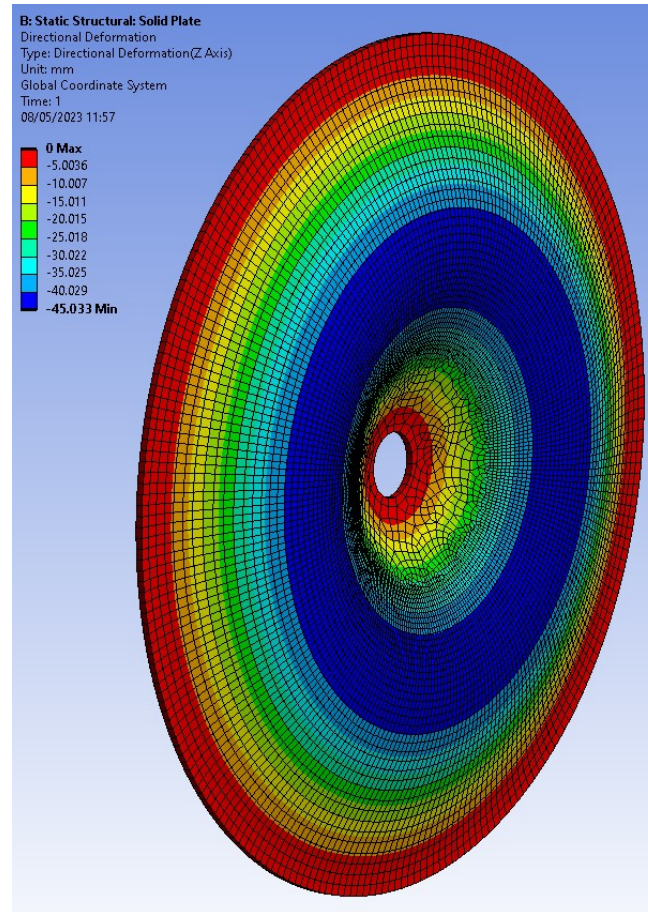
Sandwich structure



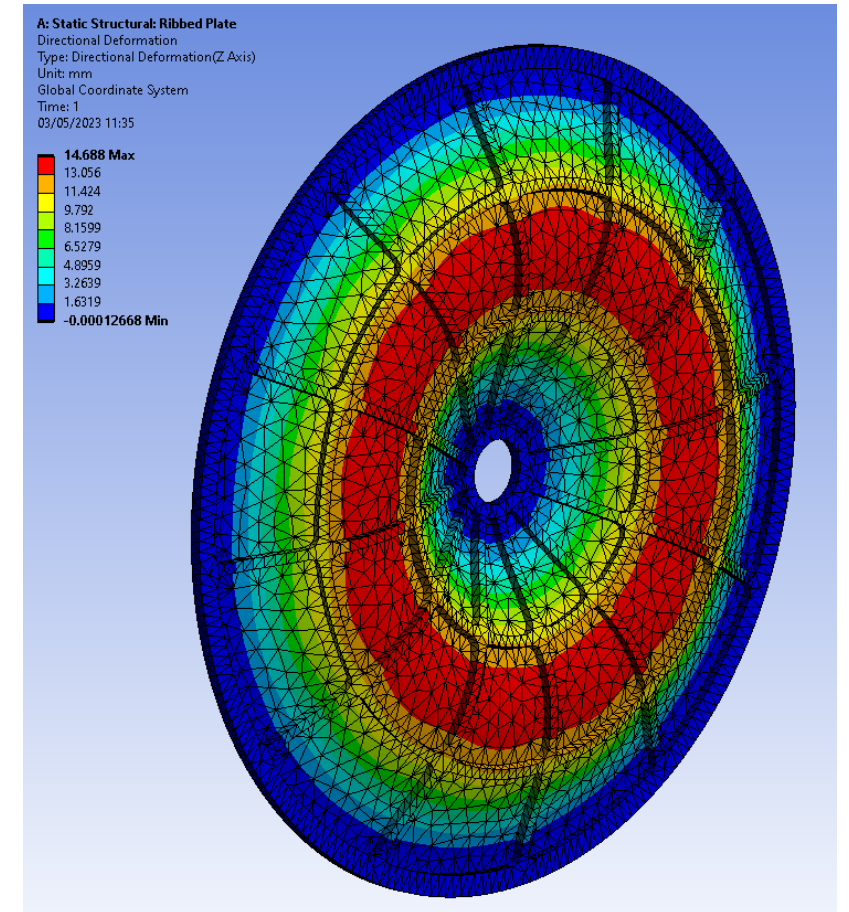
# dRICH: concept design



Solid plate  
Plate = 30 mm thick  
Material = AISI 304

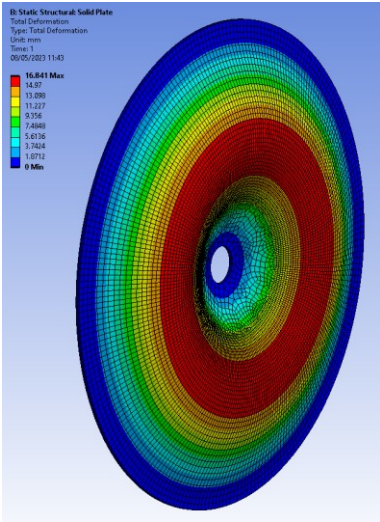


Solid plate  
Plate = 30 mm thick  
Material = Al

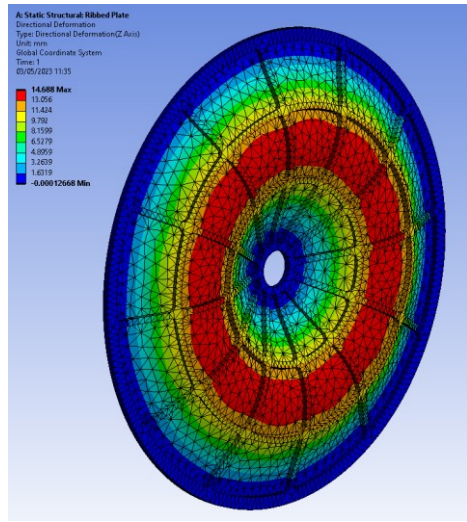


Ribbed plate  
Plate = 10 mm thick  
ribs = 50 mm thick  
Material: AISI 304

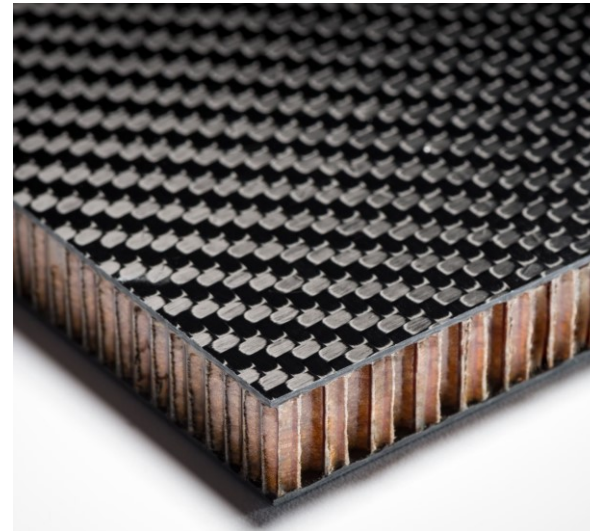
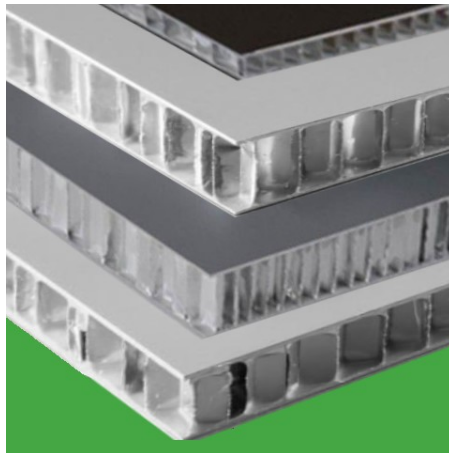
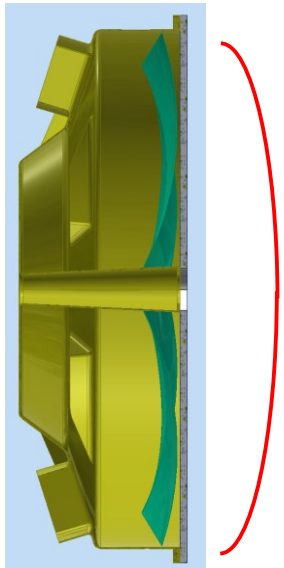
# dRICH: concept design



Sandwich plate  
Plate = 50 mm thick



Ribbed sandwich plate  
Plate = 30 mm thick  
ribs = 30 mm thick

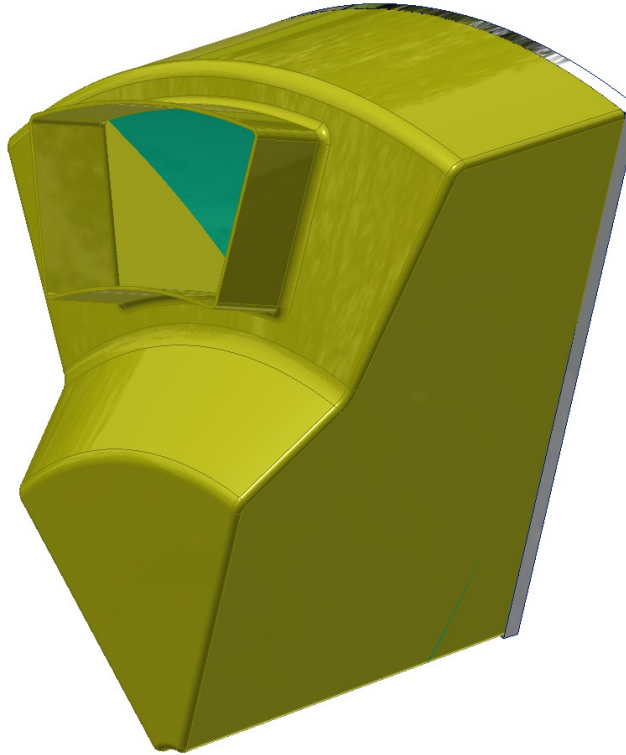


|                    | Solid Metal Sheet | Sandwich Construction        | Thicker Sandwich             |
|--------------------|-------------------|------------------------------|------------------------------|
|                    |                   |                              |                              |
| Relative Stiffness | 100               | 700<br>7 times more rigid    | 3700<br>37 times more rigid! |
| Relative Strength  | 100               | 350<br>3.5 times as strong   | 925<br>9.25 times as strong! |
| Relative Weight    | 100               | 103<br>3% increase in weight | 106<br>6% increase in weight |

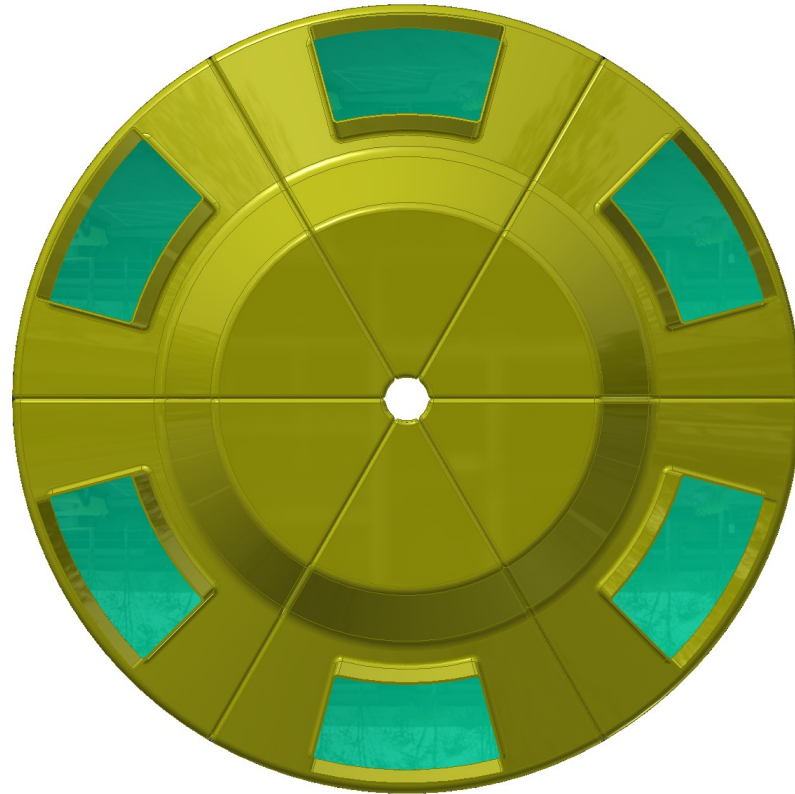
*A striking example of how honeycomb stiffens a structure without materially increasing its weight.*

Sandwich plate  
50÷120 mm thick  
2 mm Al + 46÷116 mm honeycomb + 2 mm Al

# dRICH: concept design



Single Slice



Six Slices

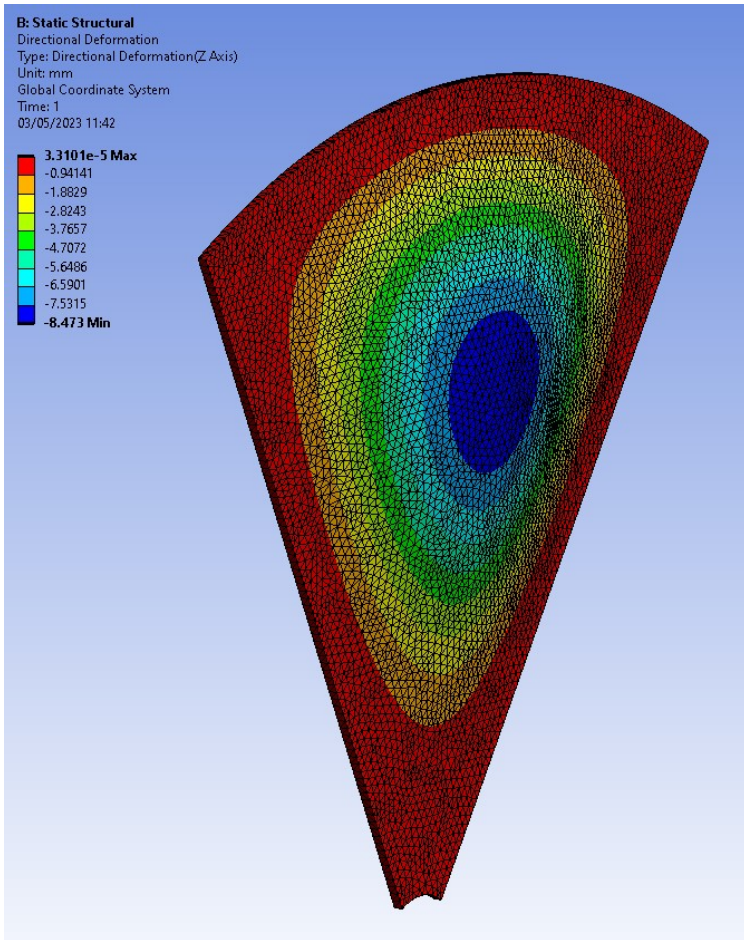
## Disadvantages

- Reduction in the active area

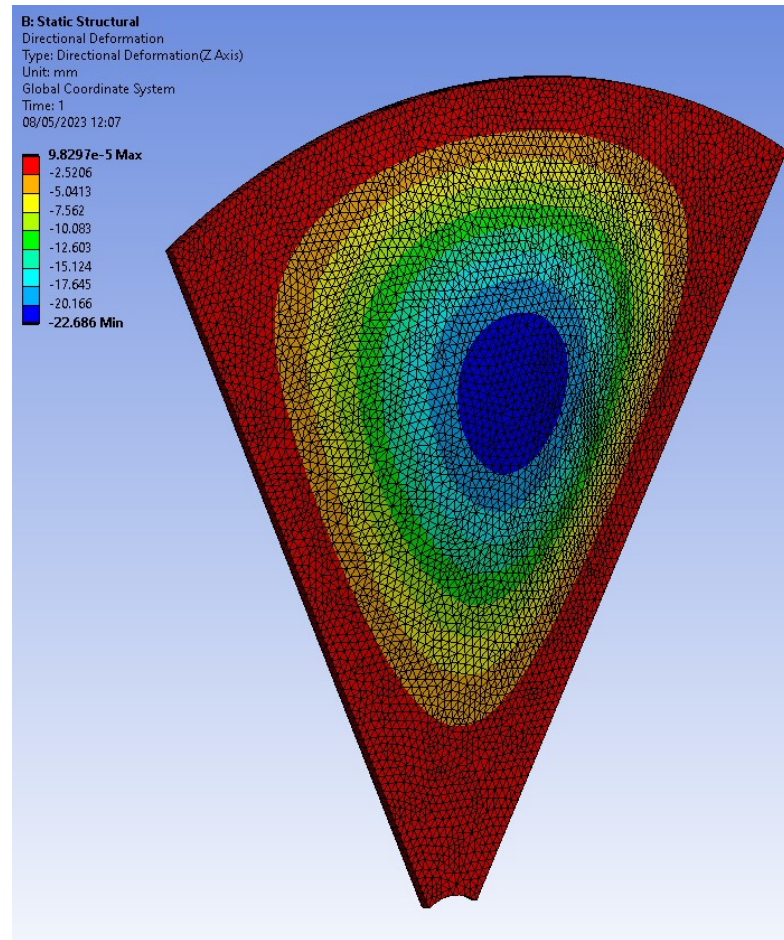
## Advantages

- Lower gas volumes to be managed
- Partitioning of the gas volume
- Reduction of thickness of components
- An important advantage is that it can be “easily” built.

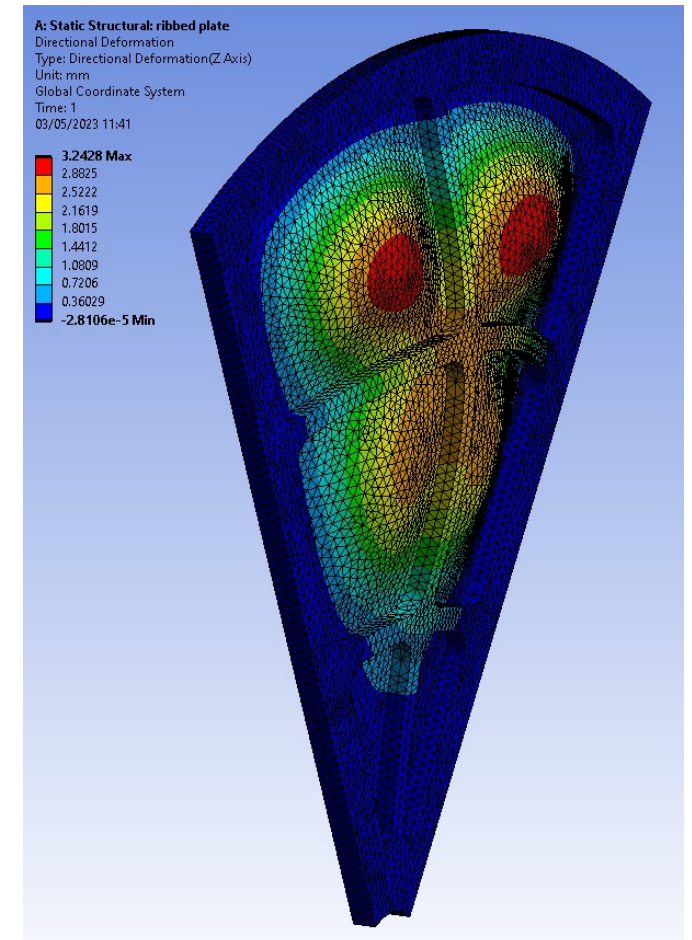
# dRICH: concept design



Solid plate  
Plate = 20 mm thick  
Material = AISI 304

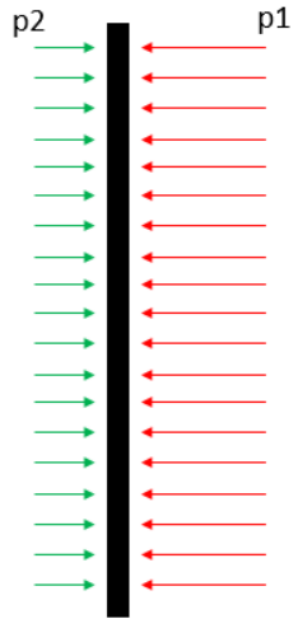
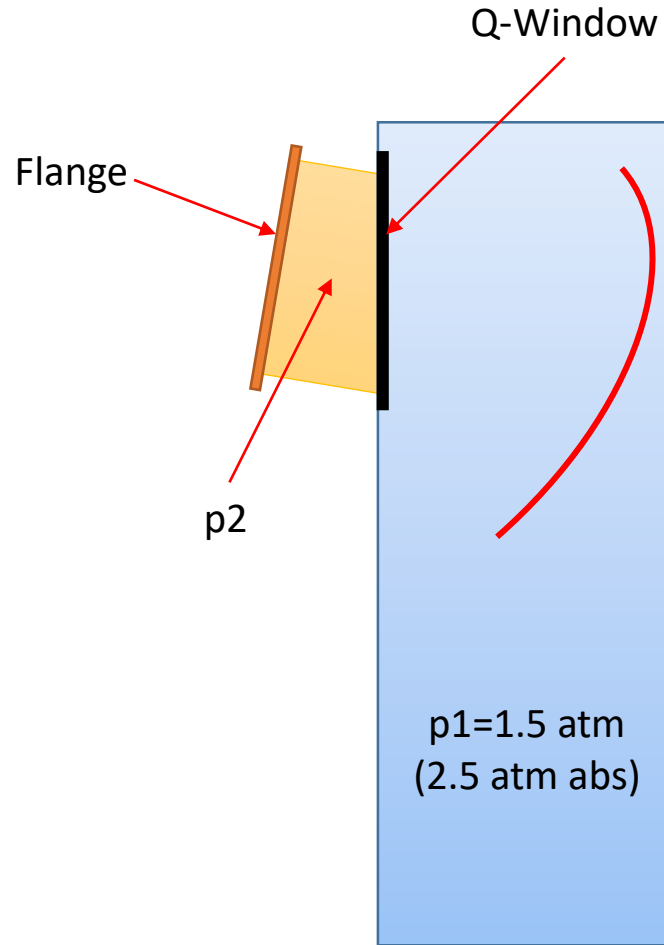


Solid plate  
Plate = 20 mm thick  
Material = Al

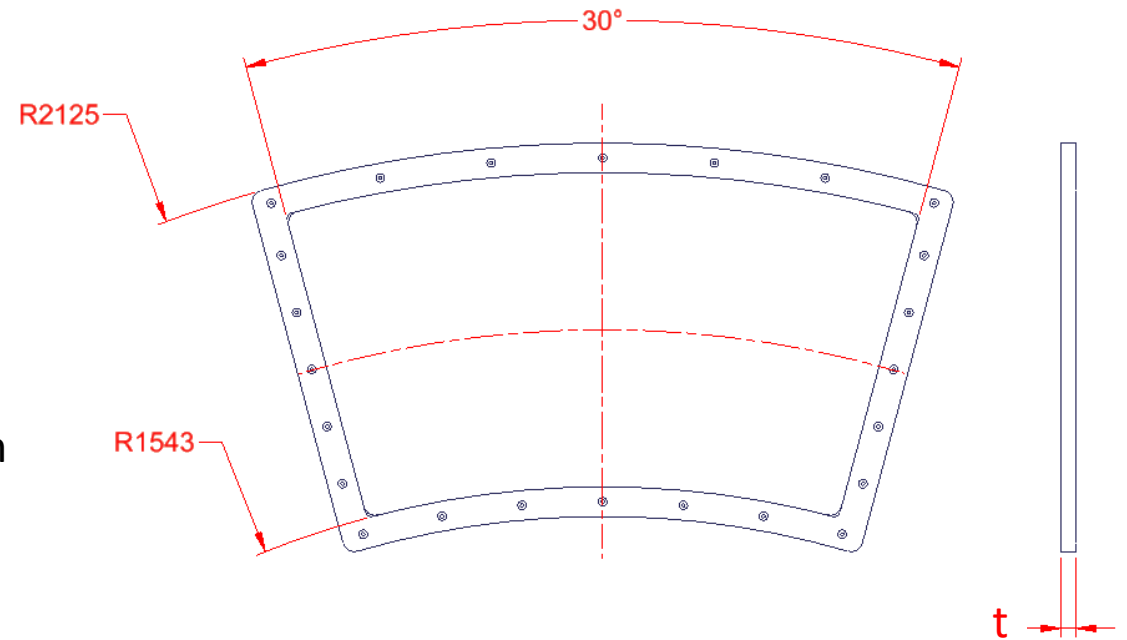


Ribbed plate  
Plate = 10 mm thick  
ribs = 50 mm thick  
Material: AISI 304

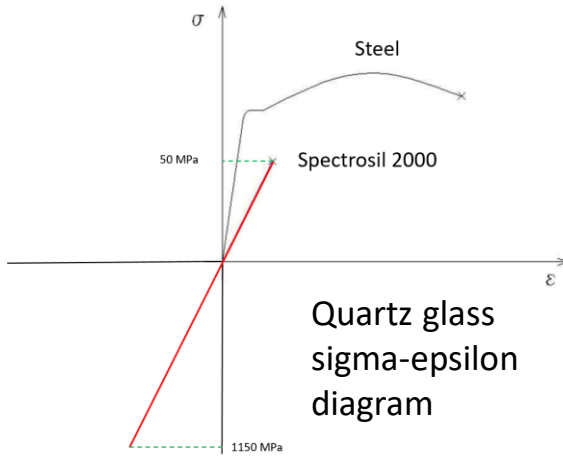
# dRICH: quartz windows



- Two scenarios:
1.  $p_2 = \text{atm press.}$
  2.  $p_2 = p_1 = 1.5 \text{ atm}$



# Quartz windows: stress and strain calculation



Brittle material

$$S = \begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}$$

$$\sigma_1 \leq \sigma_{amm,t}$$

$$-\sigma_{amm,c} \leq \sigma_3$$

Galileo-Rankine strength criterion

$$\sigma_{amm}(\text{allowable stress}) = \frac{\text{Strength of material}}{SF(\text{safety factor})}$$

$$SF(\text{brittle material}) = 2.5 \div 3$$

ASME pressure vessel code SF >= 3.5



Spectrosil 2000

| Mechanical data                    |                   | Suprasil®-family, Spectrosil®<br>Infrasil®/HOQ® |
|------------------------------------|-------------------|---|
| Density                            | g/cm <sup>3</sup> | 2,20  |
| Mohs-hardness                      |                   | 5,5.....6,5                                     |
| Micro-hardness                     | N/mm <sup>2</sup> | 8600.....9800                                   |
| Knoop-hardness                     | N/mm <sup>2</sup> | 5800.....6200                                   |
| Modulus of elasticity<br>(at 20°C) | N/mm <sup>2</sup> | 7,0 · 10 <sup>4</sup>                           |
| Modulus of torsion                 | N/mm <sup>2</sup> | 3 · 10 <sup>4</sup>                             |
| Poisson's ratio                    |                   | 0,17  |
| Compressive strength               | N/mm <sup>2</sup> | 1150  |
| Tensile strength                   | N/mm <sup>2</sup> | 50  |
| Bending strength                   | N/mm <sup>2</sup> | 67  |
| Torsional strength                 | N/mm <sup>2</sup> | 30  |
| Sound velocity                     | m/s               | 5720  |

Brittle material

$$\sigma_{amm,c} = \frac{1150}{2} \cong 575 \text{ N/mm}^2$$

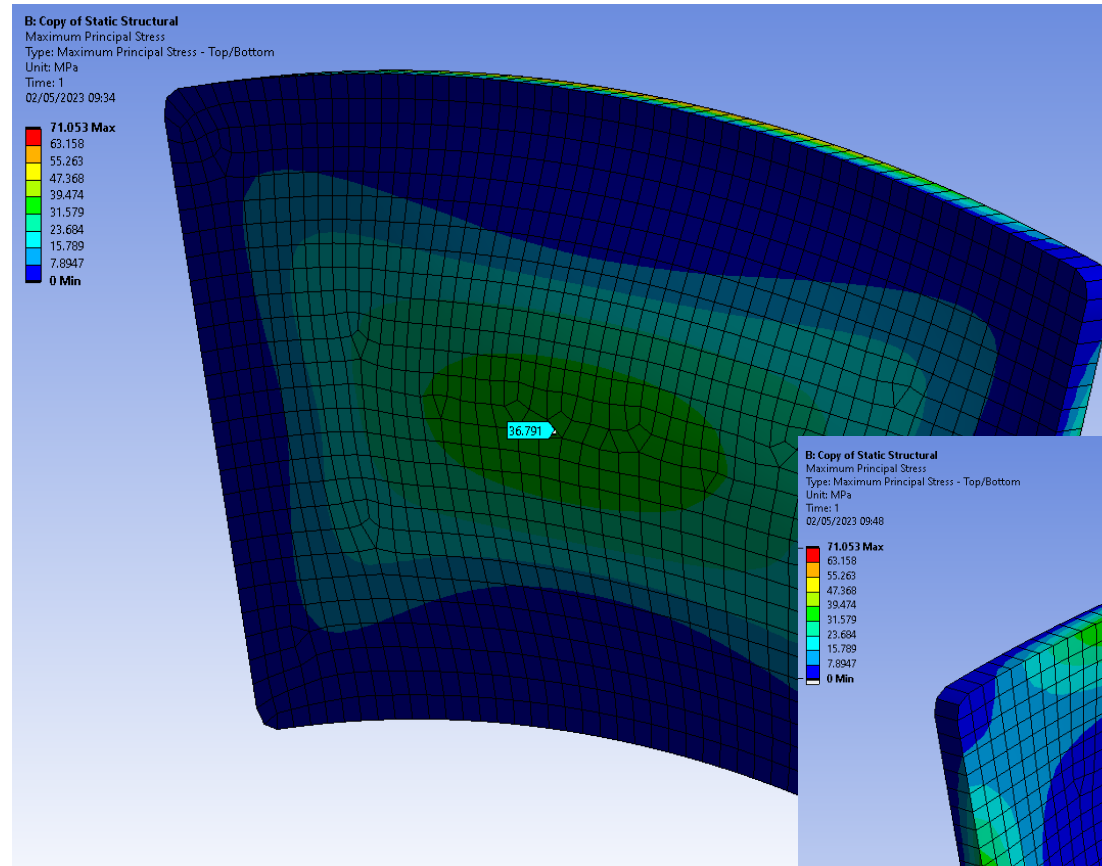
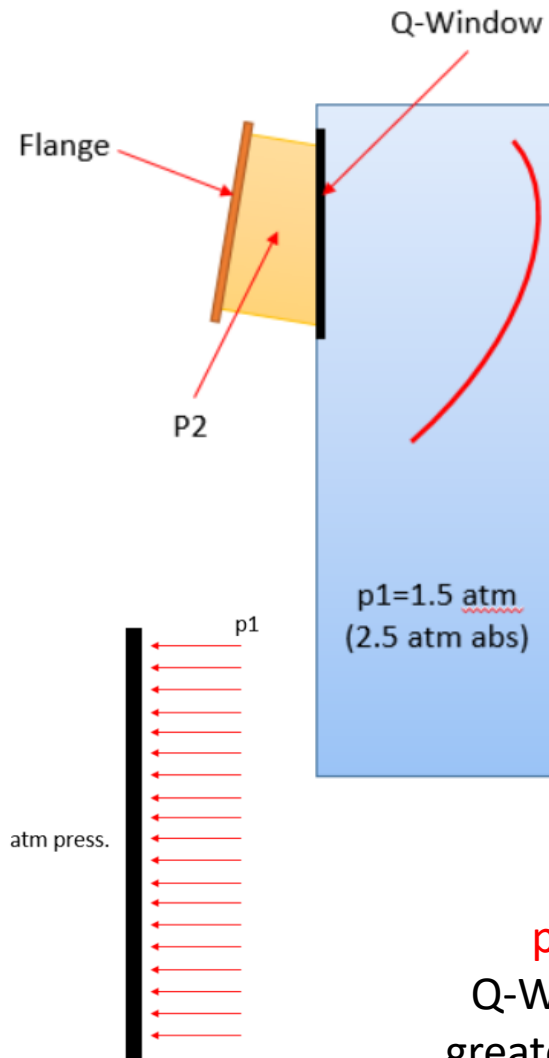
$$\sigma_{amm,c} = \frac{1150}{3.5} \cong 328 \text{ N/mm}^2$$

$$\sigma_{amm,t} = \frac{50}{2} \cong 25 \text{ N/mm}^2$$

$$\sigma_{amm,t} = \frac{50}{3.5} \cong 14 \text{ N/mm}^2$$

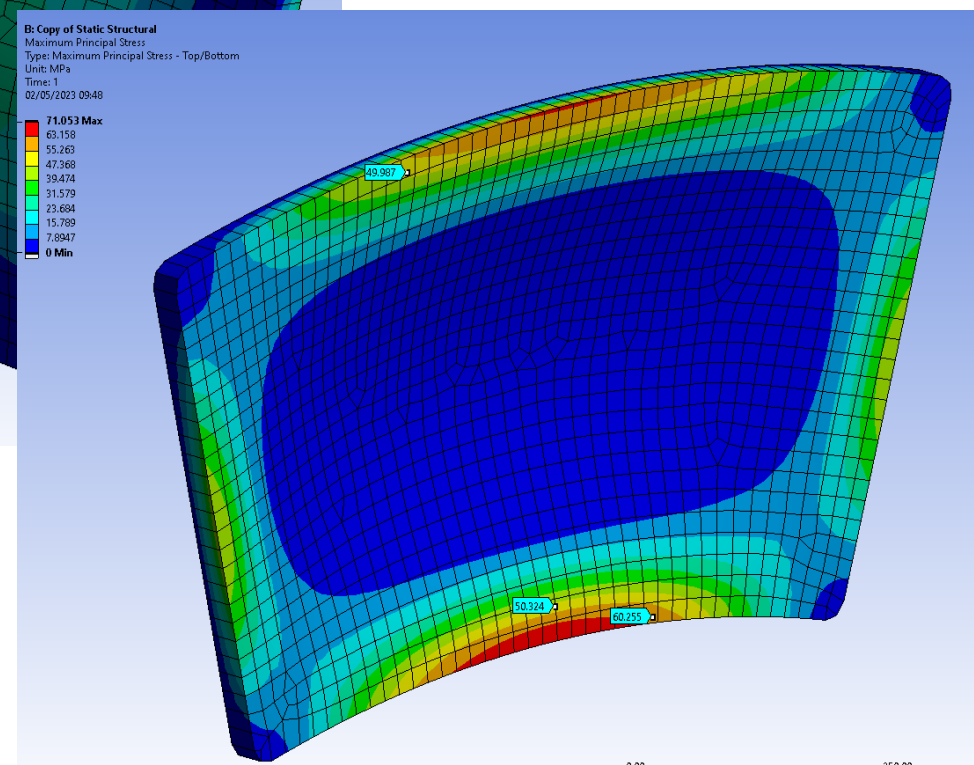


# Quartz windows FEM: scenario 1



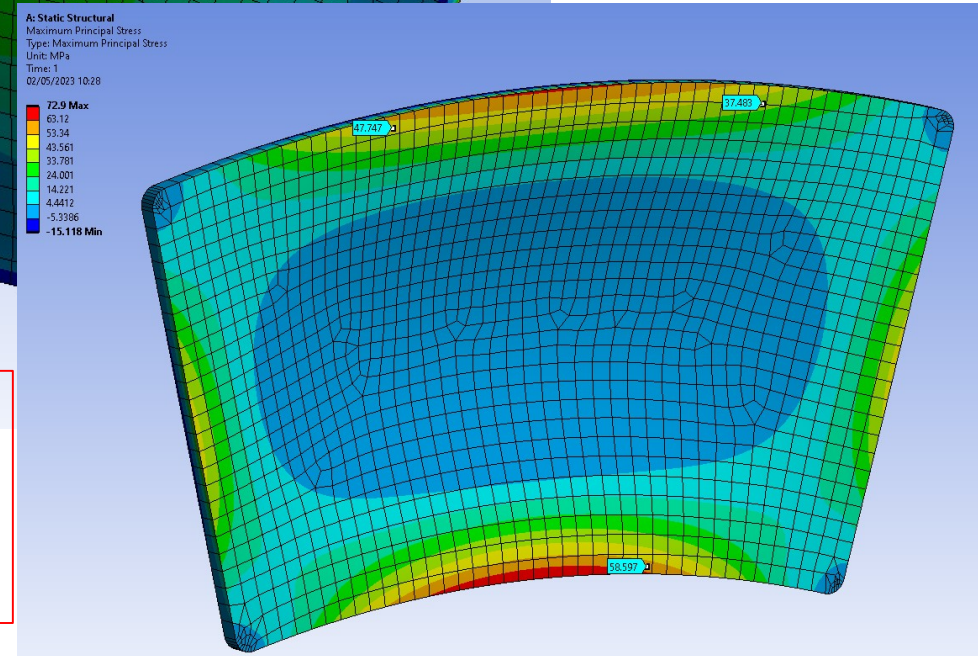
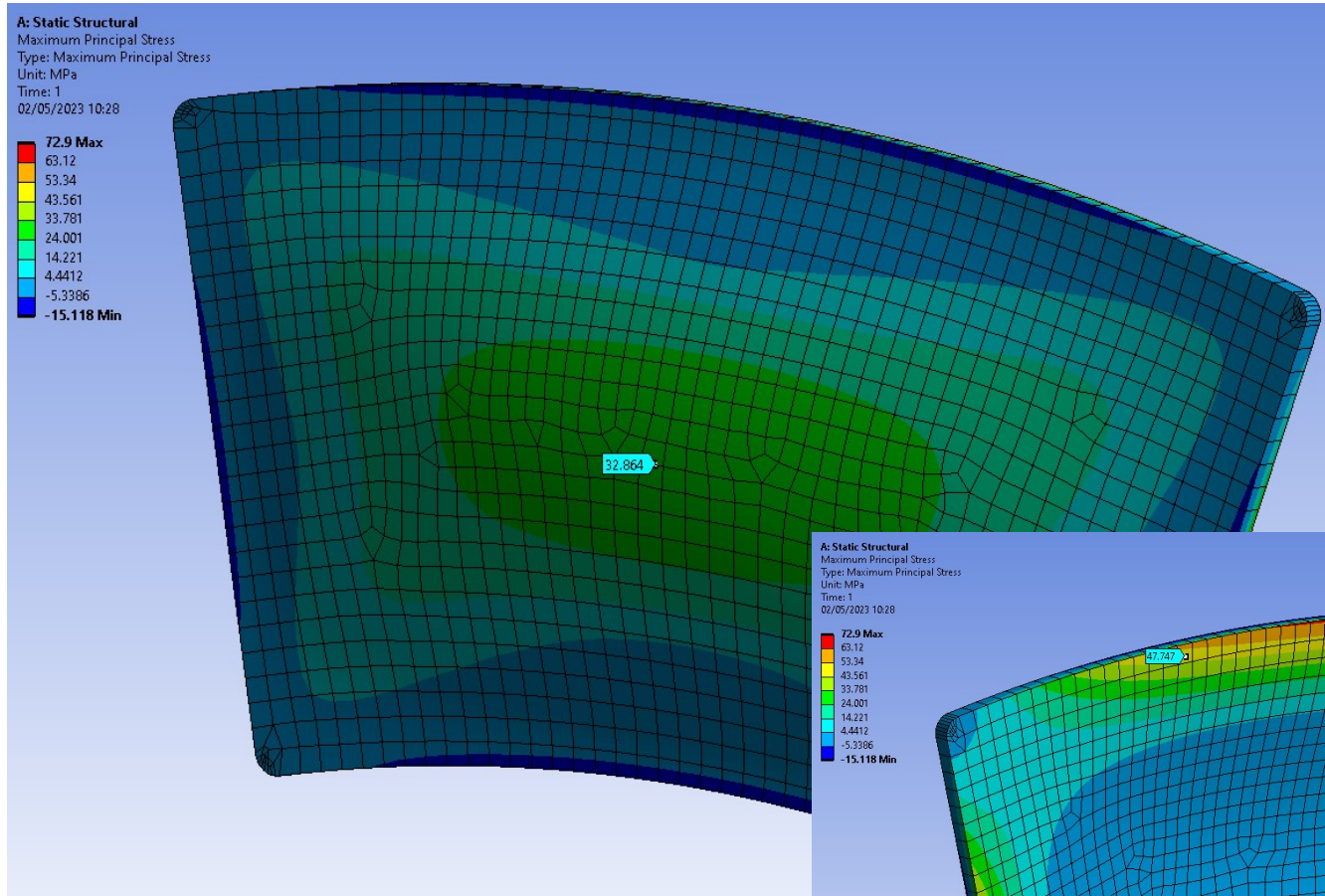
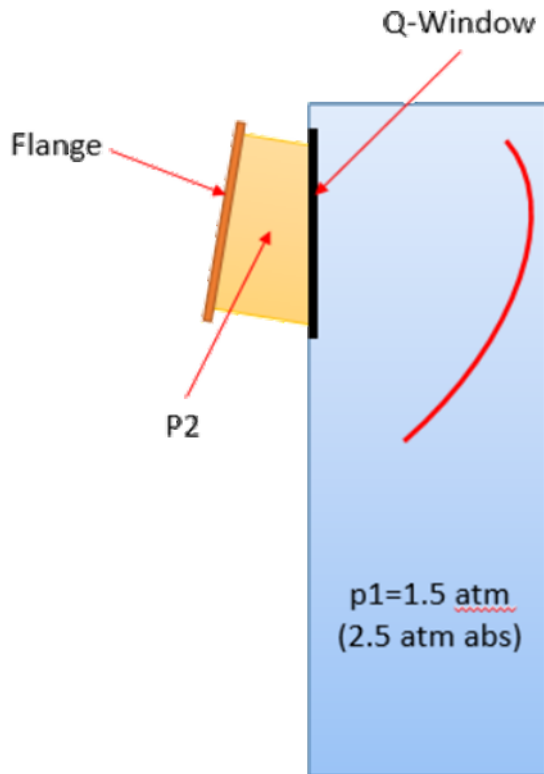
To be comply with ASME role, the Q-Window thickness should be 50/60 mm. Maybe it is incompatible with the detector optics (??)

$p1 > \text{atm pressure } (p2)$   
Q-Window thickness shall be greater than or equal to 30 mm





# Quartz windows FEM: scenario 2



$p1 = p2$

Q-Window Thickness shall be greater than or equal to 20 mm because a pressure difference inside/outside greater than 0.1 atm can lead to the breakdown of Q-W

# Quartz windows: open points

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- Effects of temperature: change of gas pressure, Q-W frame deformation
- Effects on the Q-W of main structure deformation
- Effects of pressure changes (control) during both gas enclosure filling and operation
- .....

# dRICH: critical points

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- Engineering of the gas enclosure: materials, construction technology.....
- Compliance with ASME/PED pressure vessel code
- Quartz Windows: stress
- Gas pressure control (pressure/temperature changes)



**Thank You!!**

