



Istituto Nazionale di Fisica Nucleare
SEZIONE DI FERRARA

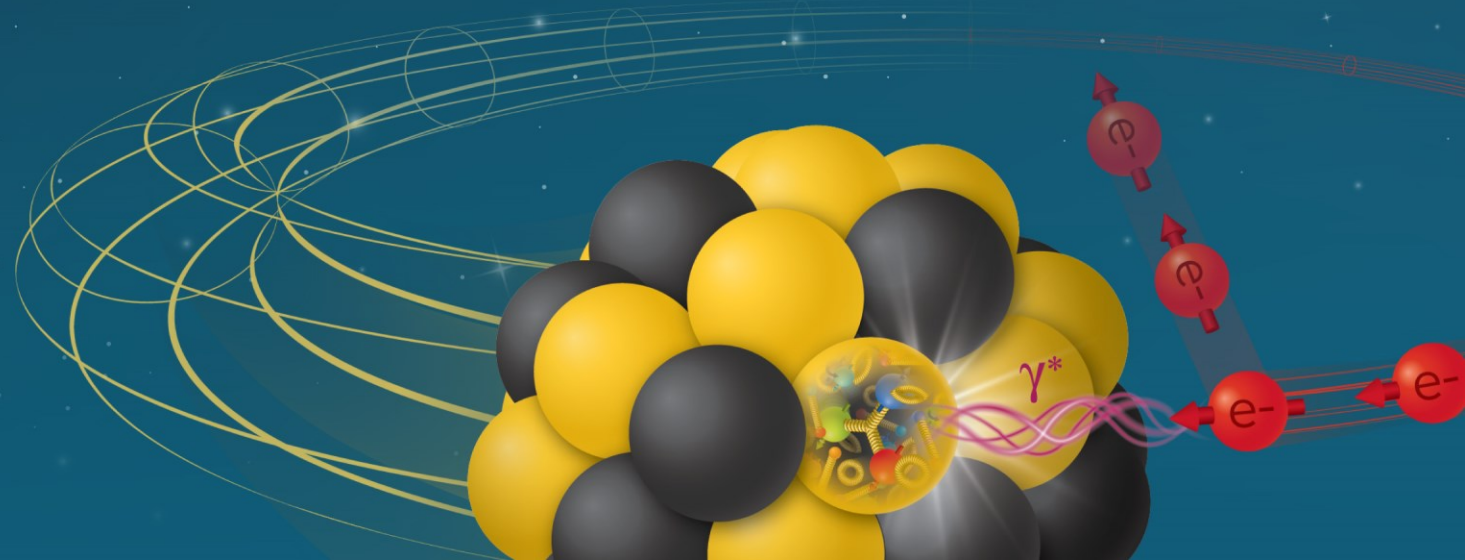


ePIC experiment dRICH

dRICH Meeting Mechanical Design Status Report

Alessandro Saputi – 19 February 2025

Electron-Ion Collider



dRICH Mechanical Design

- Main requirements: position, clearance and envelope
- Components: vessel, detector box, aerogel
- Integration

PCB Cooling

- Main requirements

Prototype

- Design

Target R&D and Generic R&D

- Spectrometer Chamber
- High Pressure Chamber

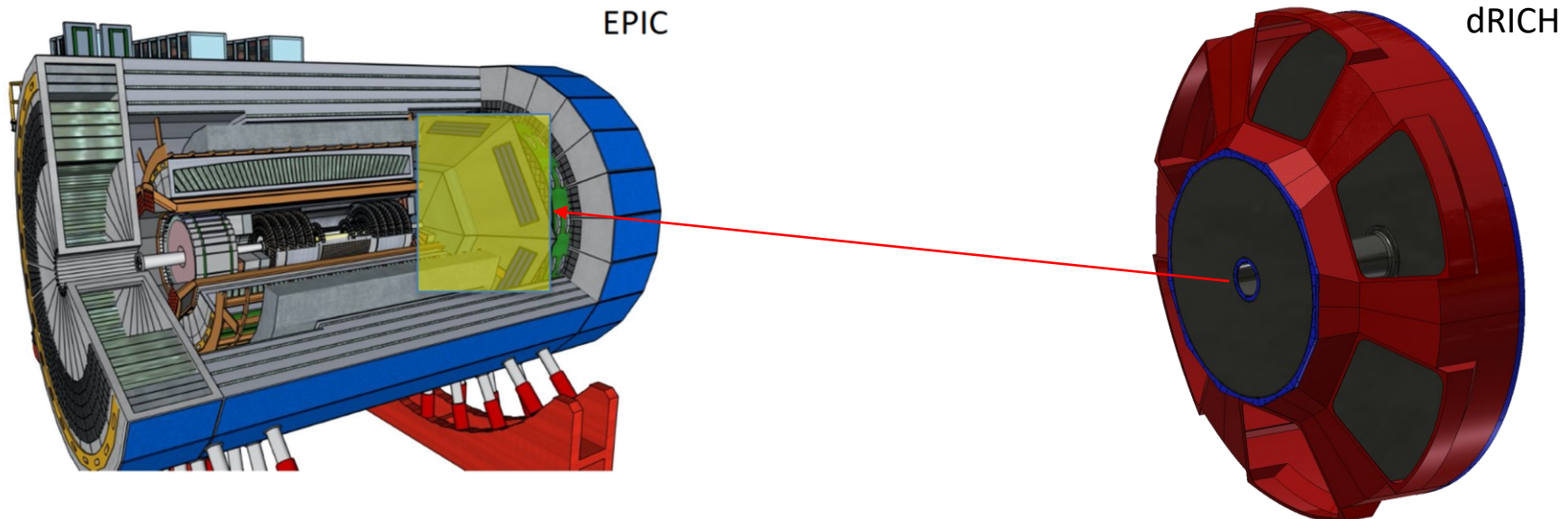
-
- Technical drawing of the IP 3800 door. The drawing shows a side view of the door with its dimensions. The overall width is 1980 mm, and the overall height is 3800 mm. The door is divided into two main sections, each 1270 mm wide. The drawing includes a detailed view of the door's profile and a small inset showing the door's handle and lock mechanism.



dRICH: main requirements

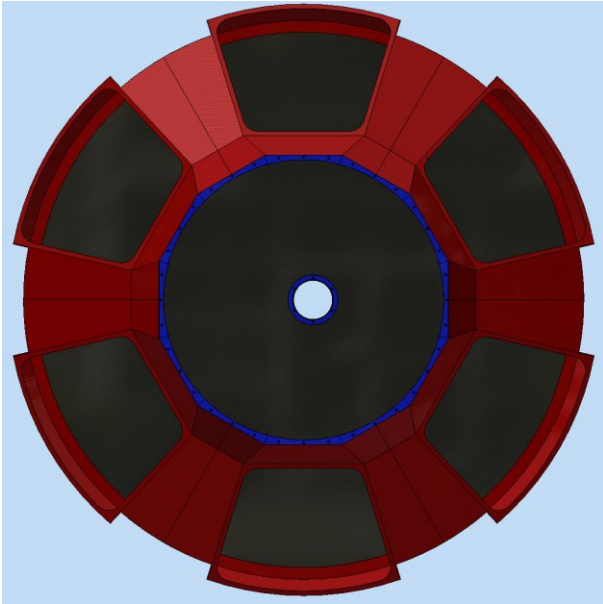
The major functions of the dRICH mechanical structure (gas enclosure) are to provide containment for the dRICH gas radiator and to act as a stable frame for the optical components (the mirrors and aerogel):

- It must be light-tight.
- It must ensure the stability of the structure under the influence of the magnetic field.
- The enclosure must withstand a differential pressure of 3-10 mbar without compromising the mirror alignment.
- The minimum amount of material must be placed within the ePIC experiment acceptance limits.

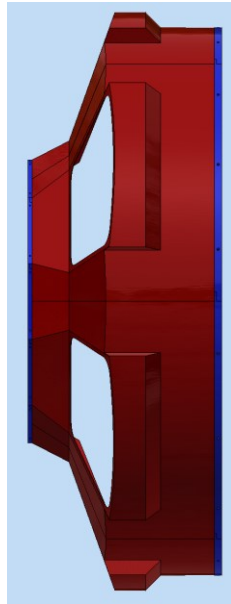


dRICH: mechanical preliminary design

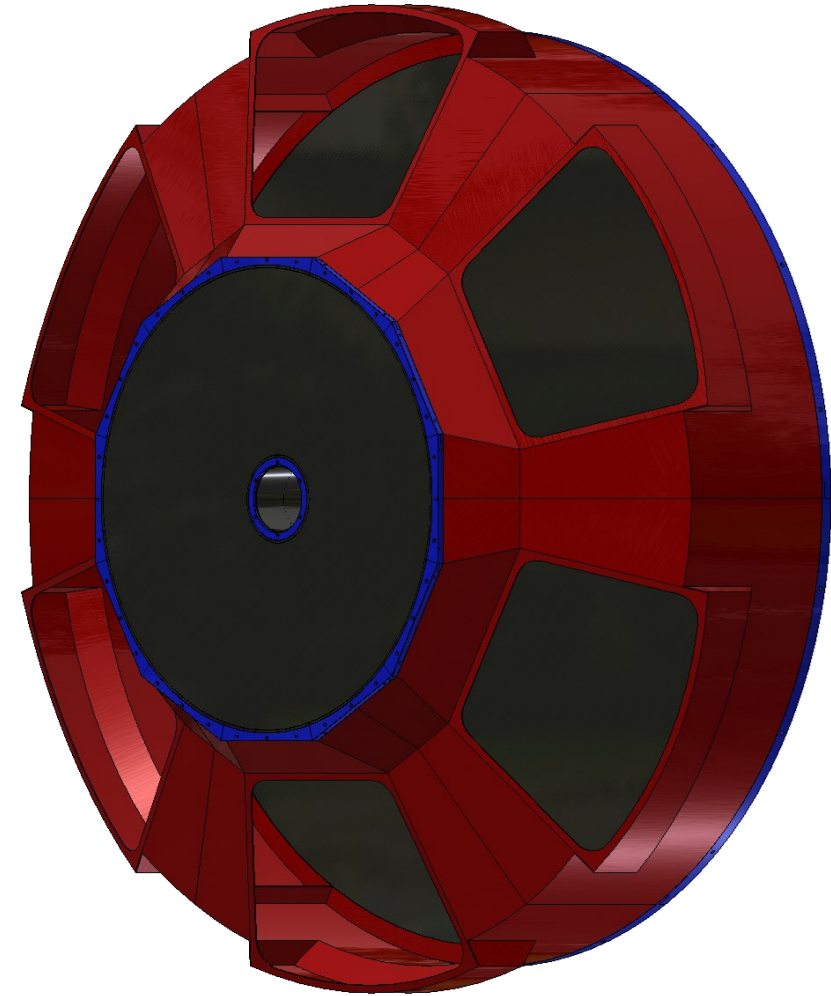
Front view



Side view

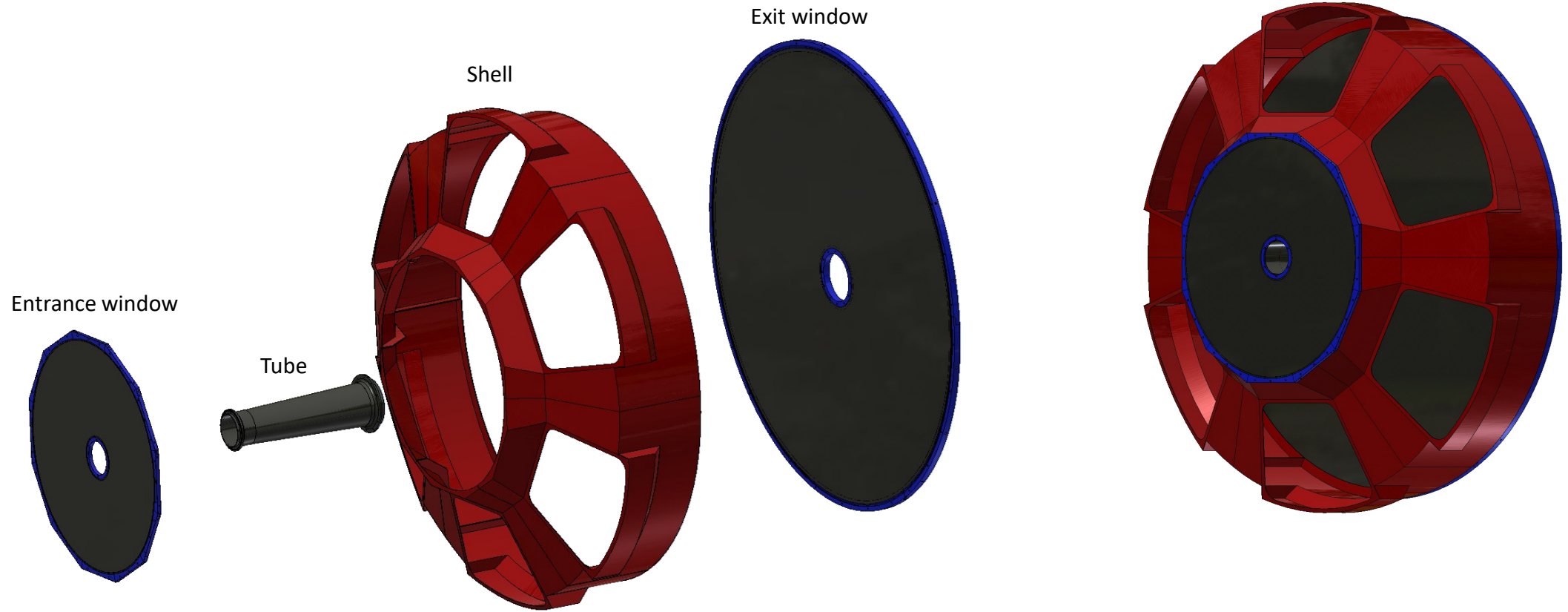


Back view



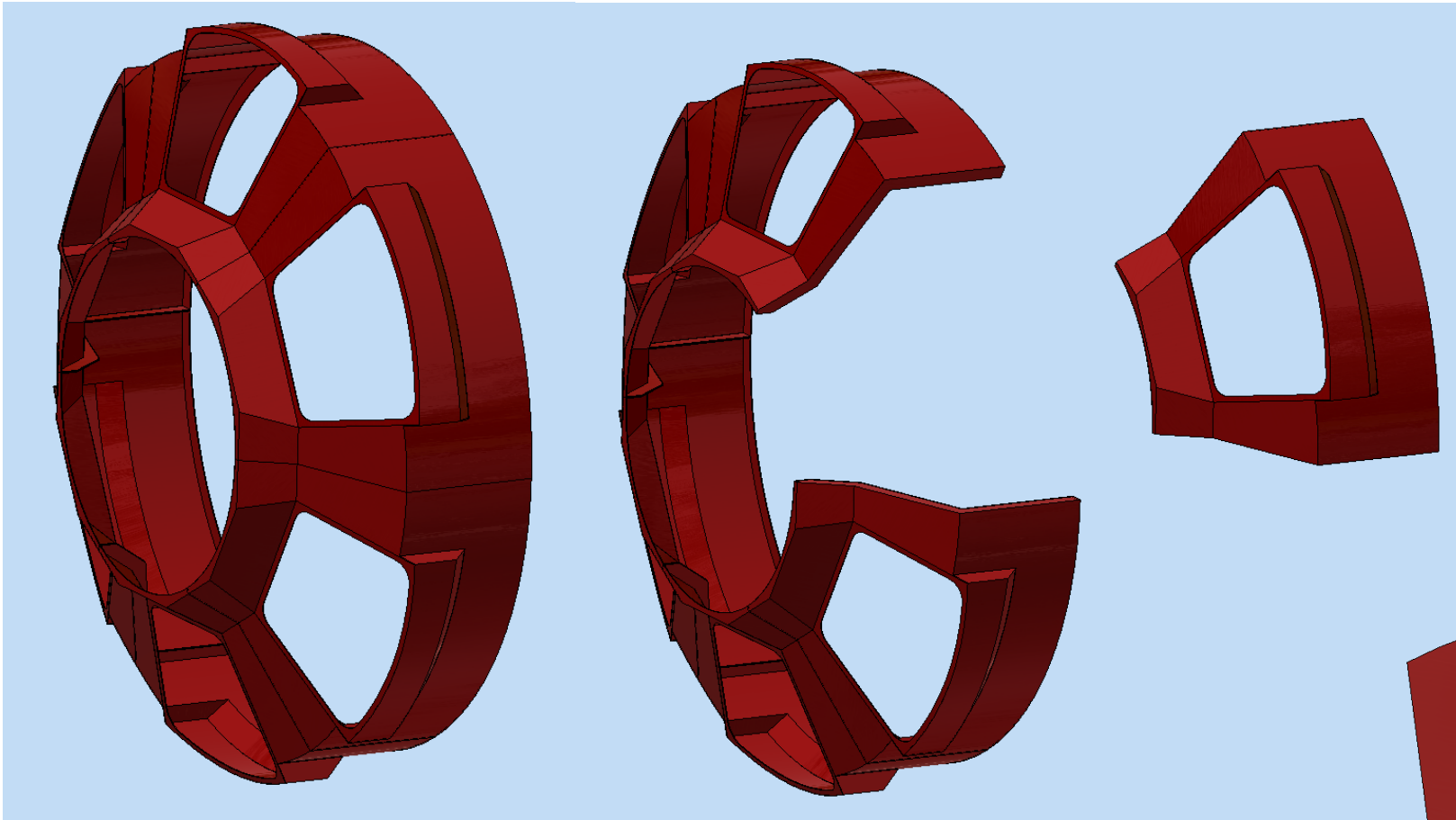
The gas enclosure is essentially a cylindrical box that hosts PDUs (including the cooling system), quartz windows, mirrors, and aerogel tiles.

dRICH: mechanical structure (gas enclosure)



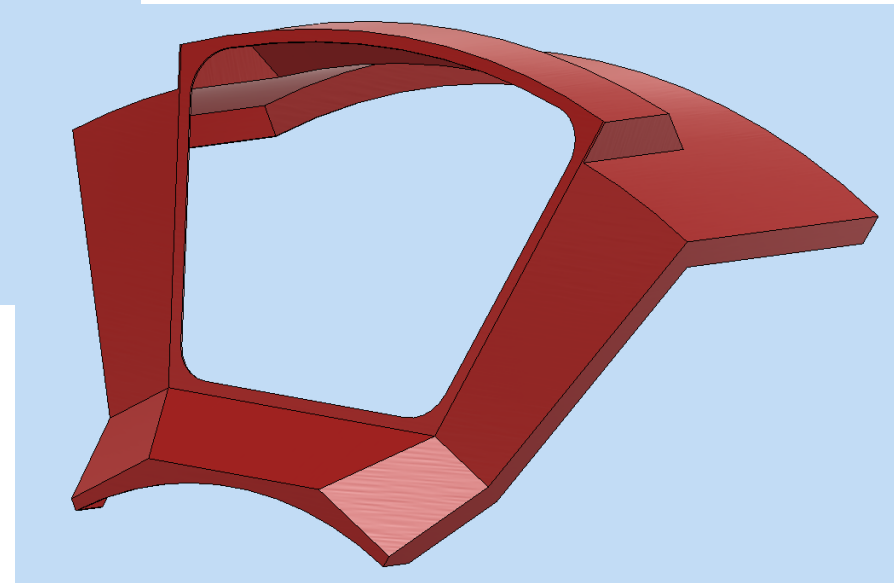
The mechanical structure (gas enclosure) is composed of six main components: the shell, the entrance and exit windows, the central tube, the detector boxes, and the quartz windows.

dRICH: shell

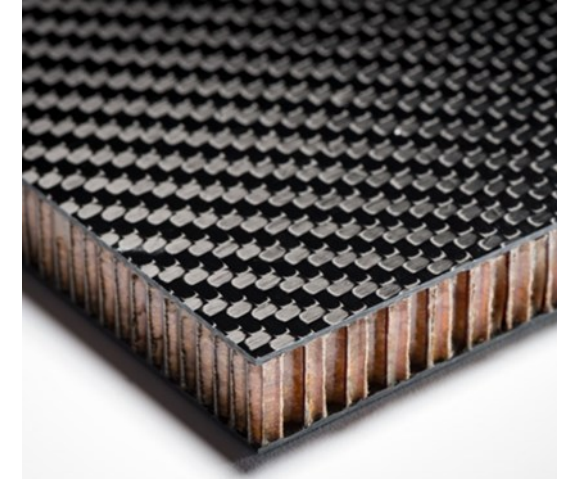
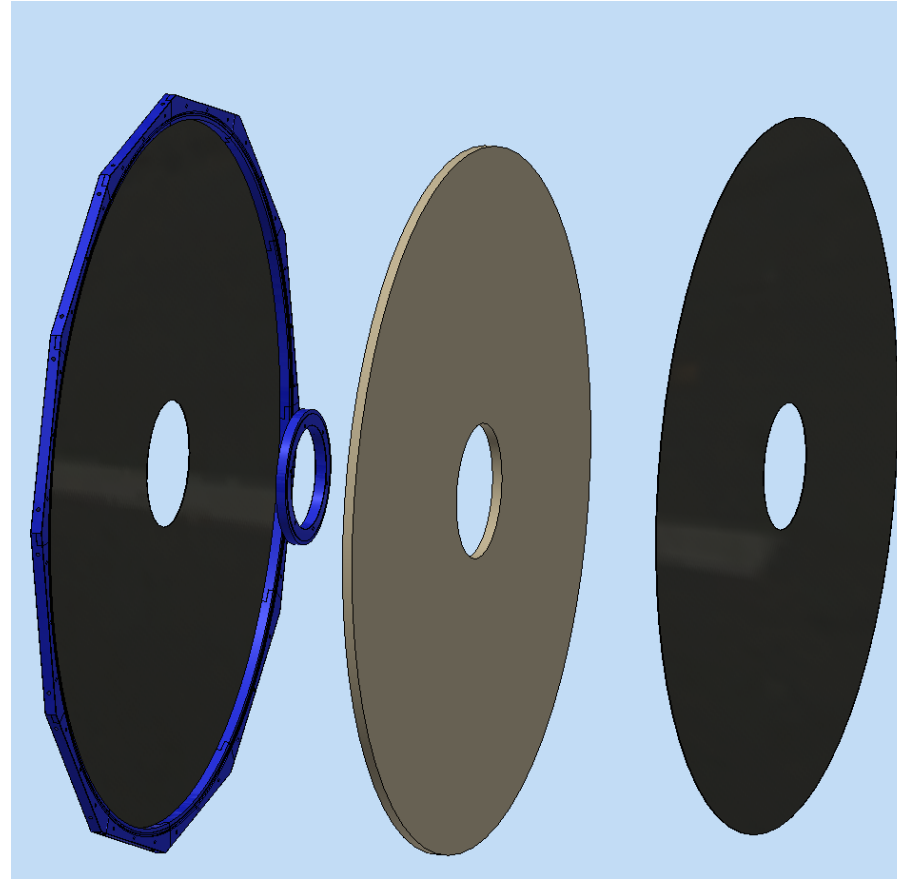
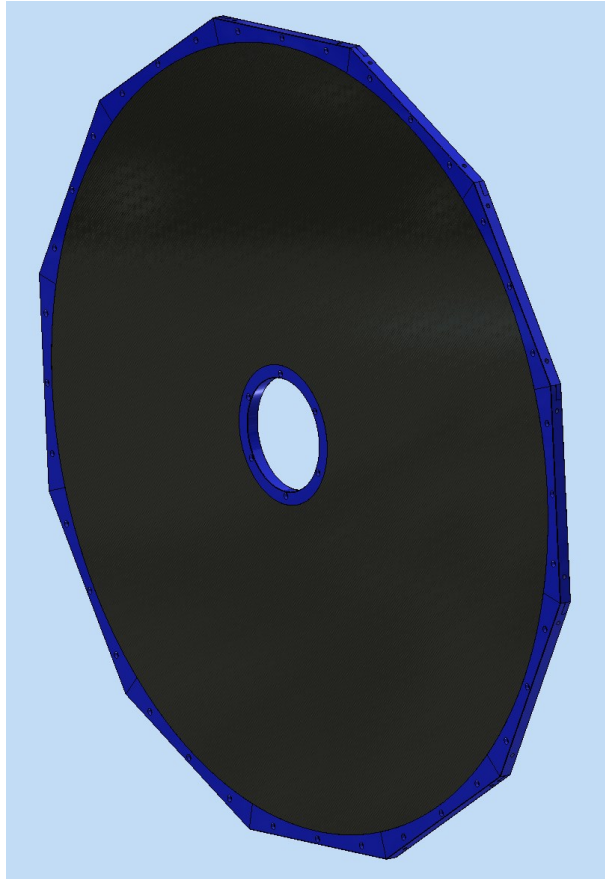


The shell will be made of an 10 mm thick carbon fibre epoxy composite. Each laminate will consist of six layers of balanced weave fabric, with fibres oriented at $0^\circ/90^\circ$ in one layer and $\pm 45^\circ$ in the adjacent layer.

The shell is composed of six parts that are both bolted and glued together to ensure structural integrity.



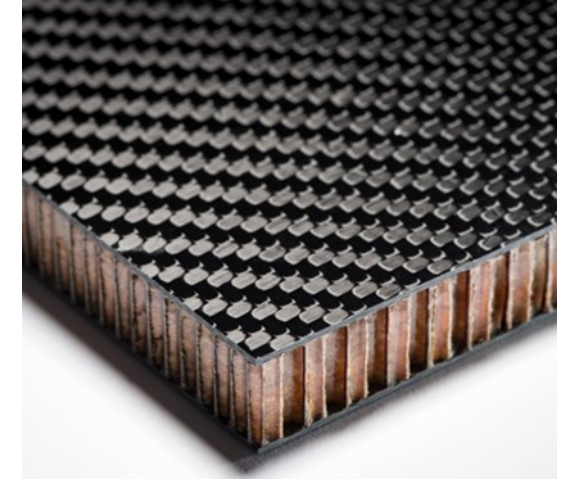
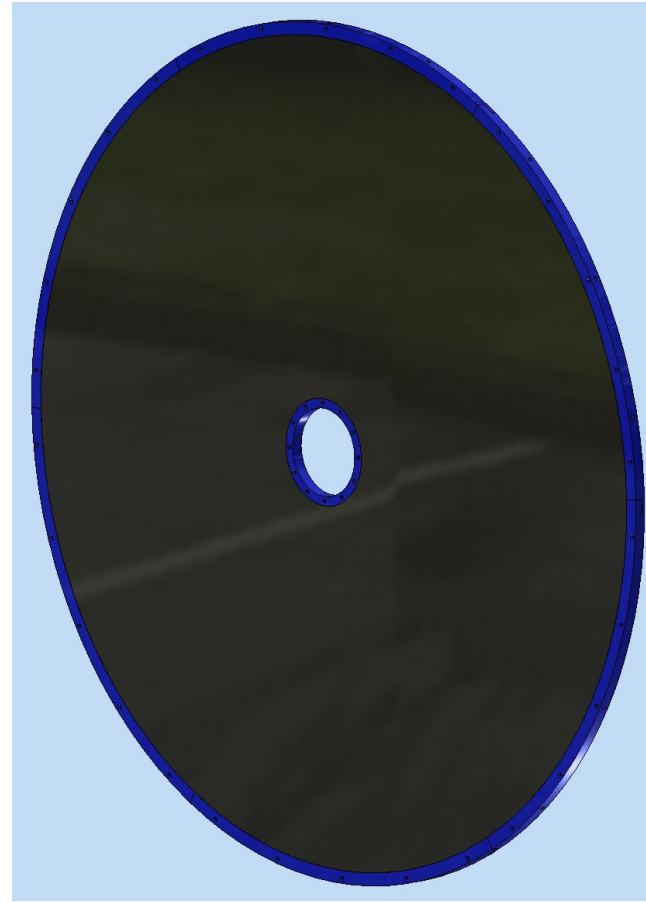
dRICH: entrance windows



The Entrance Window will be a sandwich panel consisting of two carbon fiber-reinforced epoxy skins, each 2.28 mm thick, separated by a 25 mm thick Nomex honeycomb core. Each skin is composed of six layers of balanced weave laminate, with fibers oriented at $0^\circ/90^\circ$ in one layer and overlapped with $\pm 45^\circ$ in the adjacent layer.

The external sides are enclosed by two solid frames made of carbon fiber (CF) or aluminum.

dRICH: exit windows



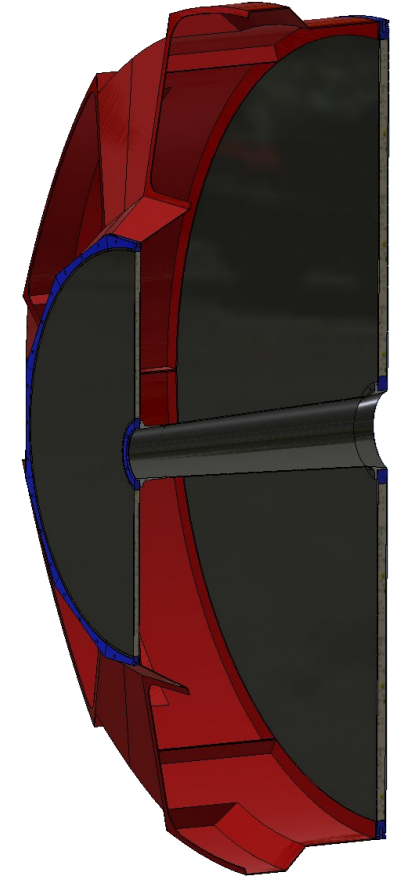
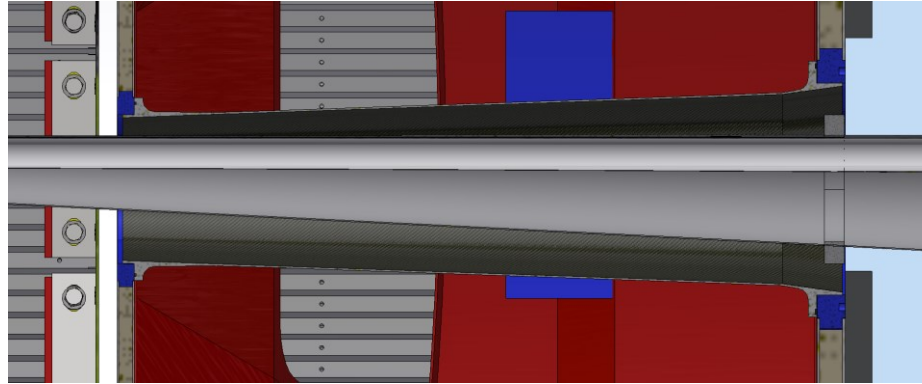
The Exit Window will be a sandwich panel consisting of two carbon fiber-reinforced epoxy skins, each 4.56 mm thick, separated by a 40 mm thick Nomex honeycomb core. Each skin is composed of six layers of balanced weave laminate, with fibers oriented at $0^\circ/90^\circ$ in one layer and overlapped with $\pm 45^\circ$ in the adjacent layer.

The external sides are enclosed by two solid frames made of carbon fiber (CF) or aluminum.

dRICH: central tube

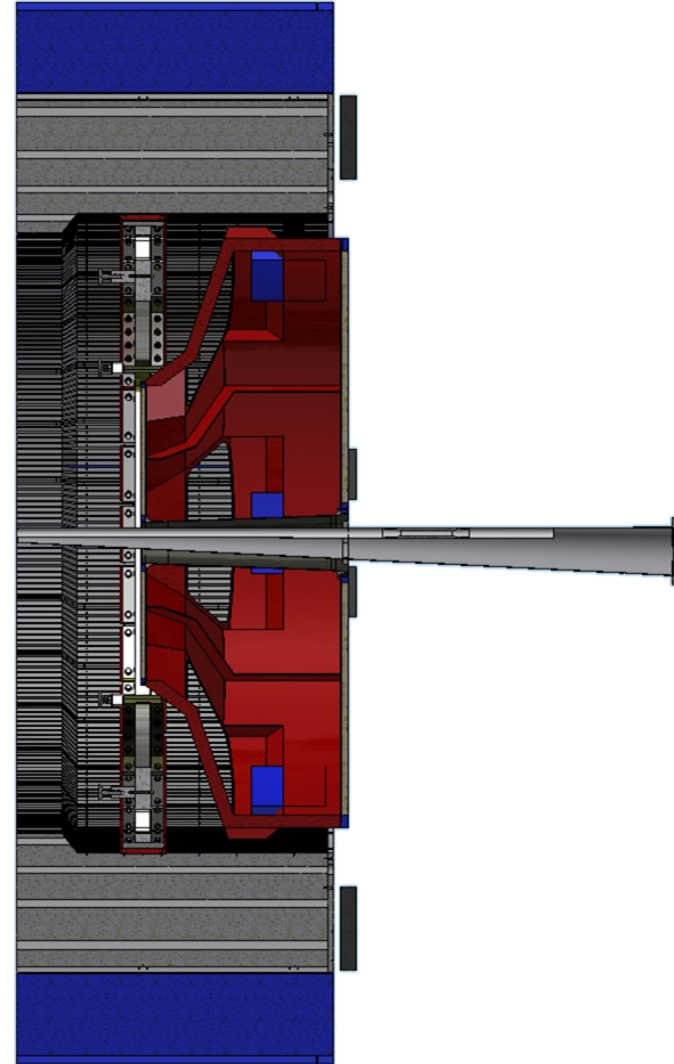
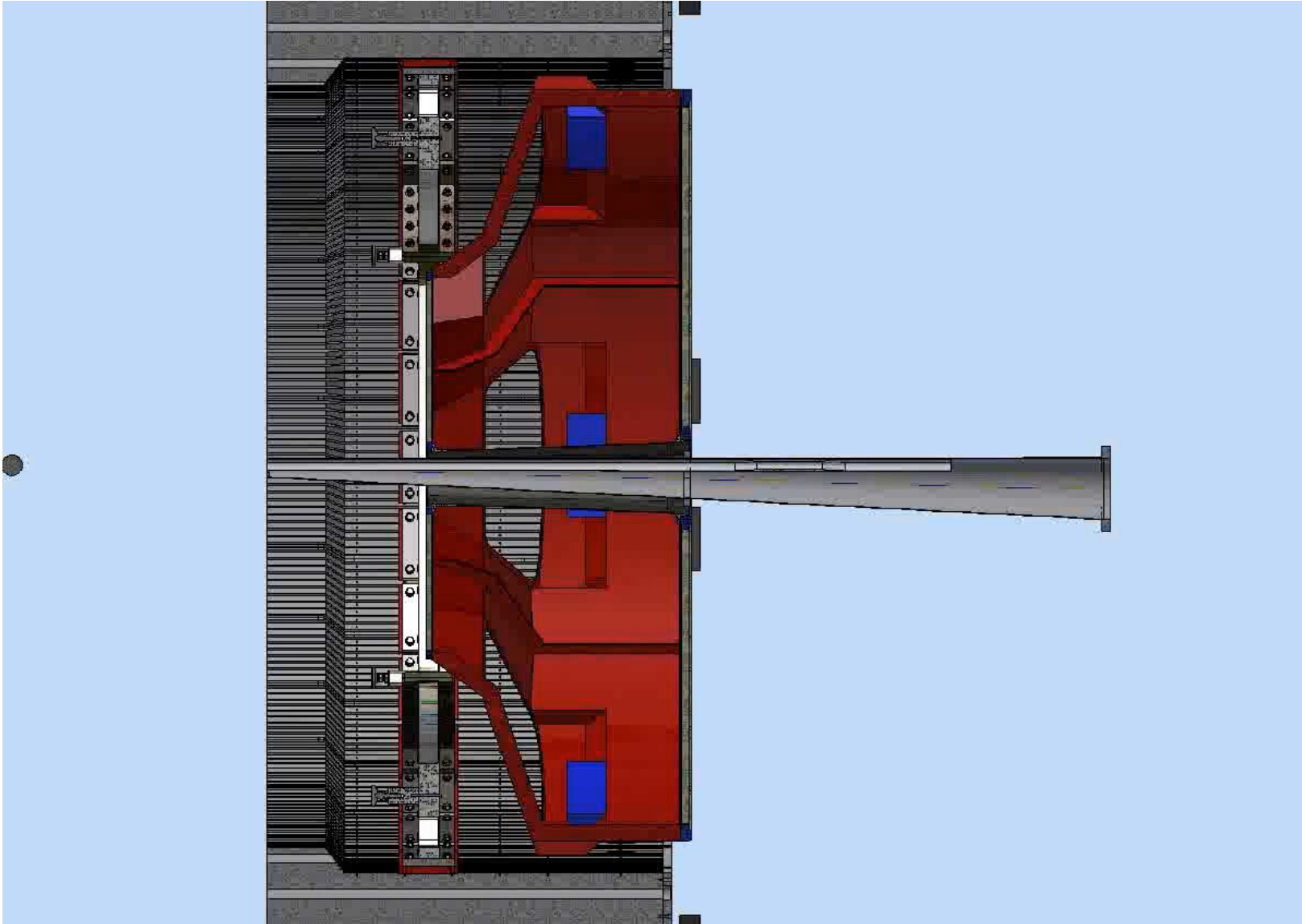


Central tube



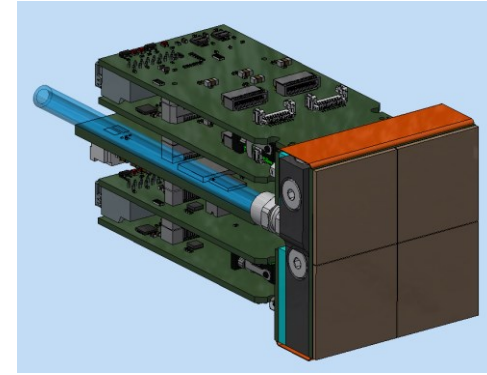
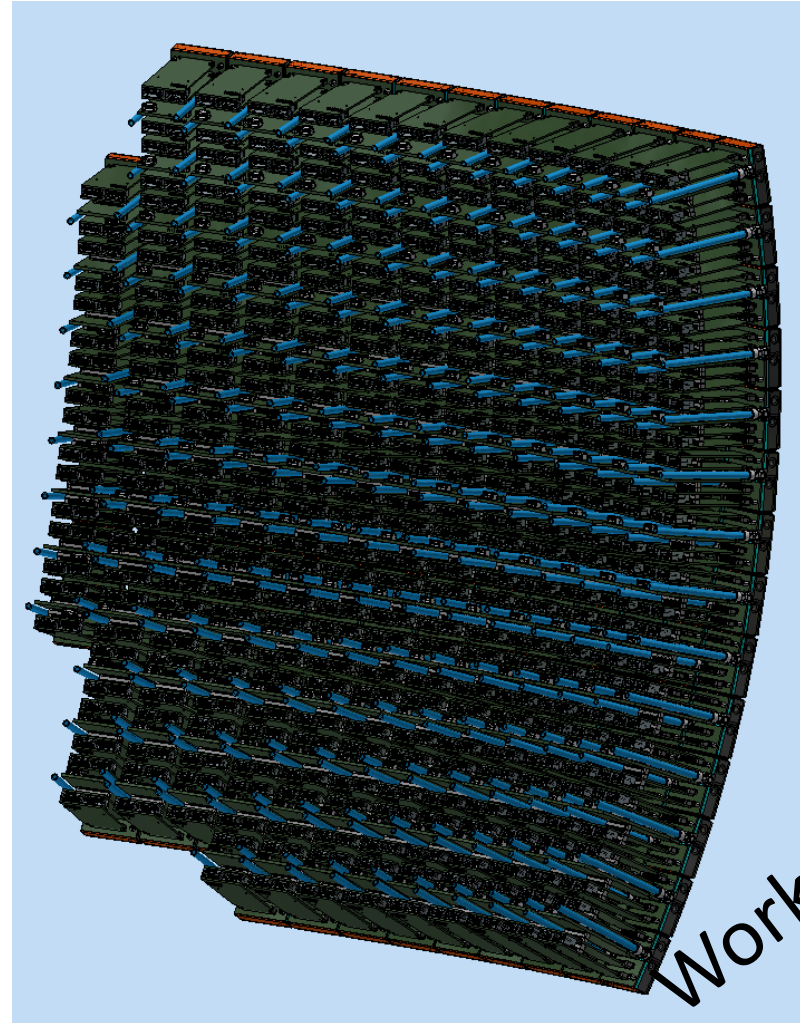
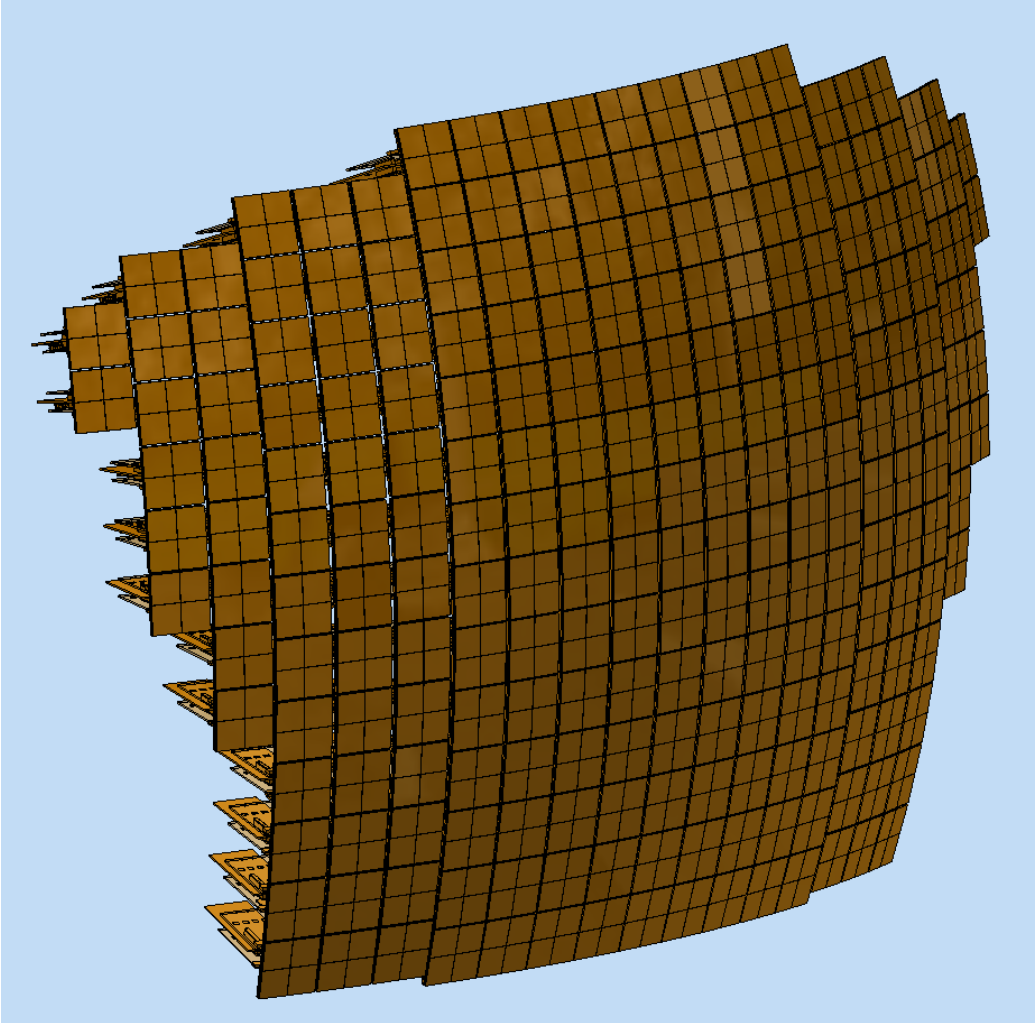
Both the entrance and exit windows are connected by the central tube. The central tube will be made of a 5 mm thick carbon fiber epoxy composite and will have an inside diameter of 260 mm at the entrance window, tapering to 370 mm at the exit window. This design ensures a radial separation between the vacuum chamber and the central tube.

dRICH: integration



dRICH: detector and detector box

The detector is composed of 236 PDUs arranged on a sphere with a radius of 1100 mm



PDU designed by
R.Preghenella

Work in progress

dRICH: aerogel layout

Aerogel Density $\approx 0.15 \text{ g/cm}^3$.

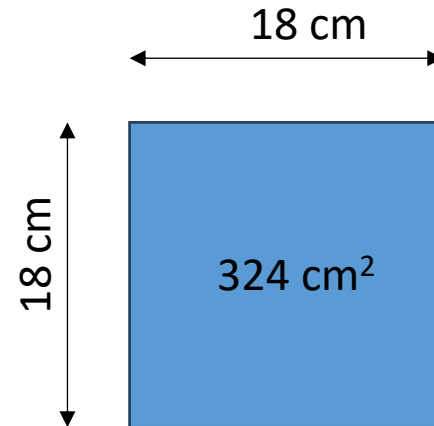
Tile size = 180 mm x 180 mm - 20 mm thick

Requirements:

Aerogel Total Thickness 40 mm

Minimize dead space between the aerogel tiles

Light-tight material on the edges of the aerogel tiles



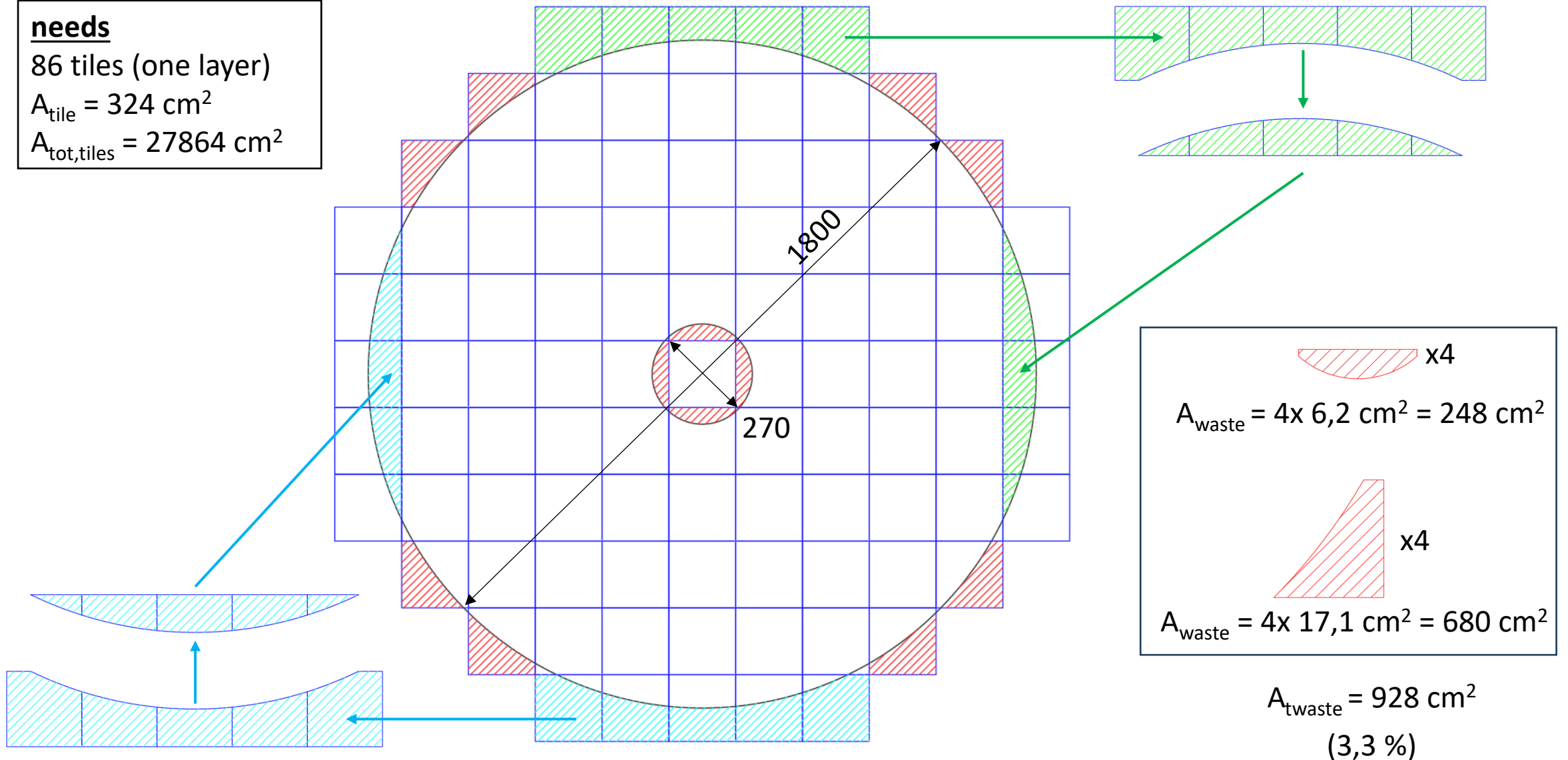
AEROGEL – Layout_A: nesting_A

needs

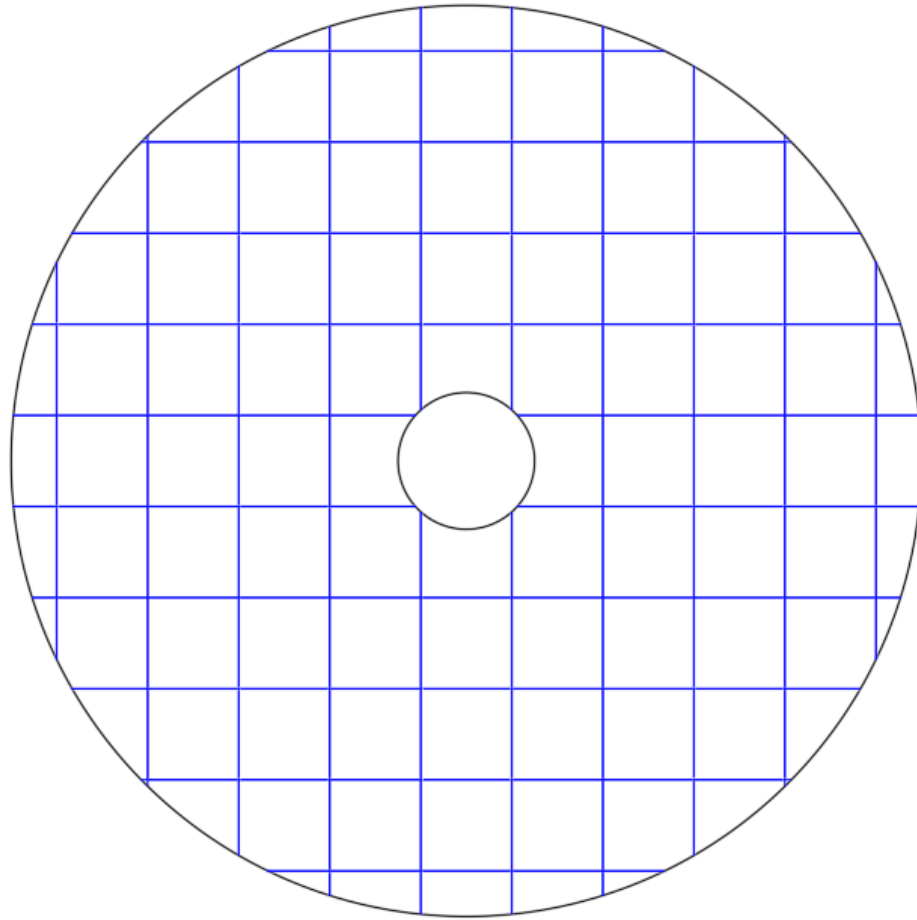
86 tiles (one layer)

$$A_{\text{tile}} = 324 \text{ cm}^2$$

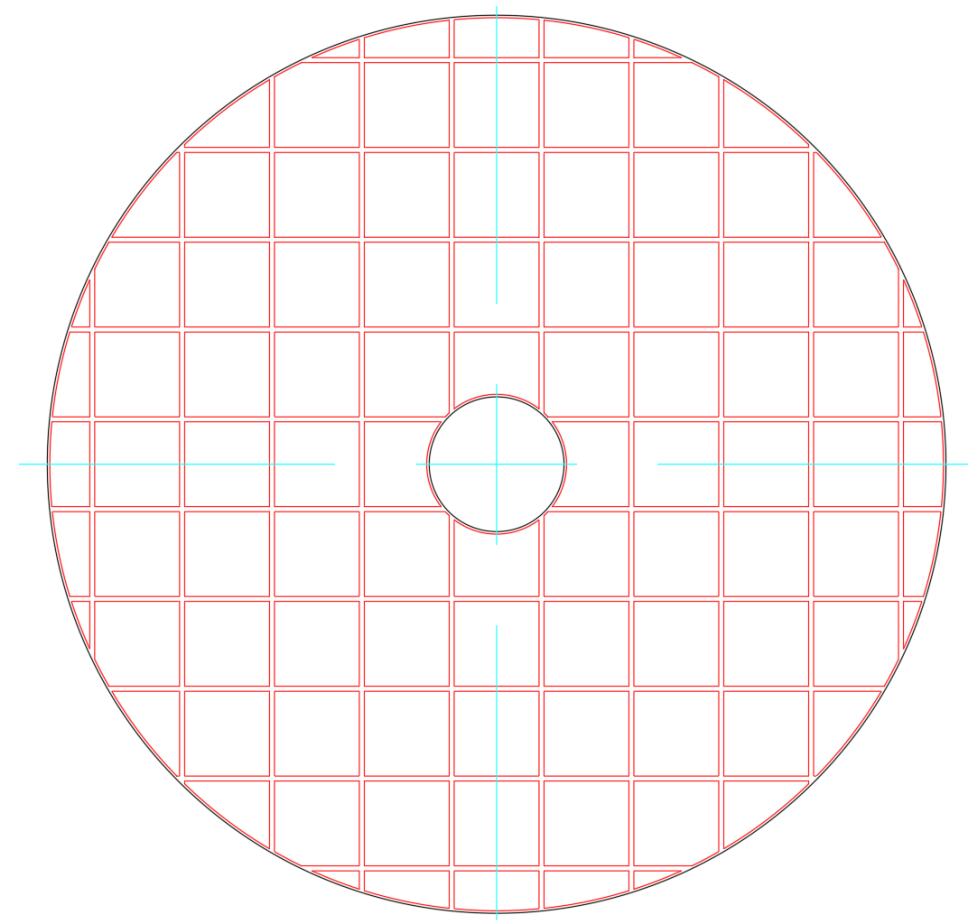
$$A_{\text{tot,tiles}} = 27864 \text{ cm}^2$$



AEROGEL – Layout_A: Active Area and Dead Space



Nominal Active Area (A_n) = 24874 cm²

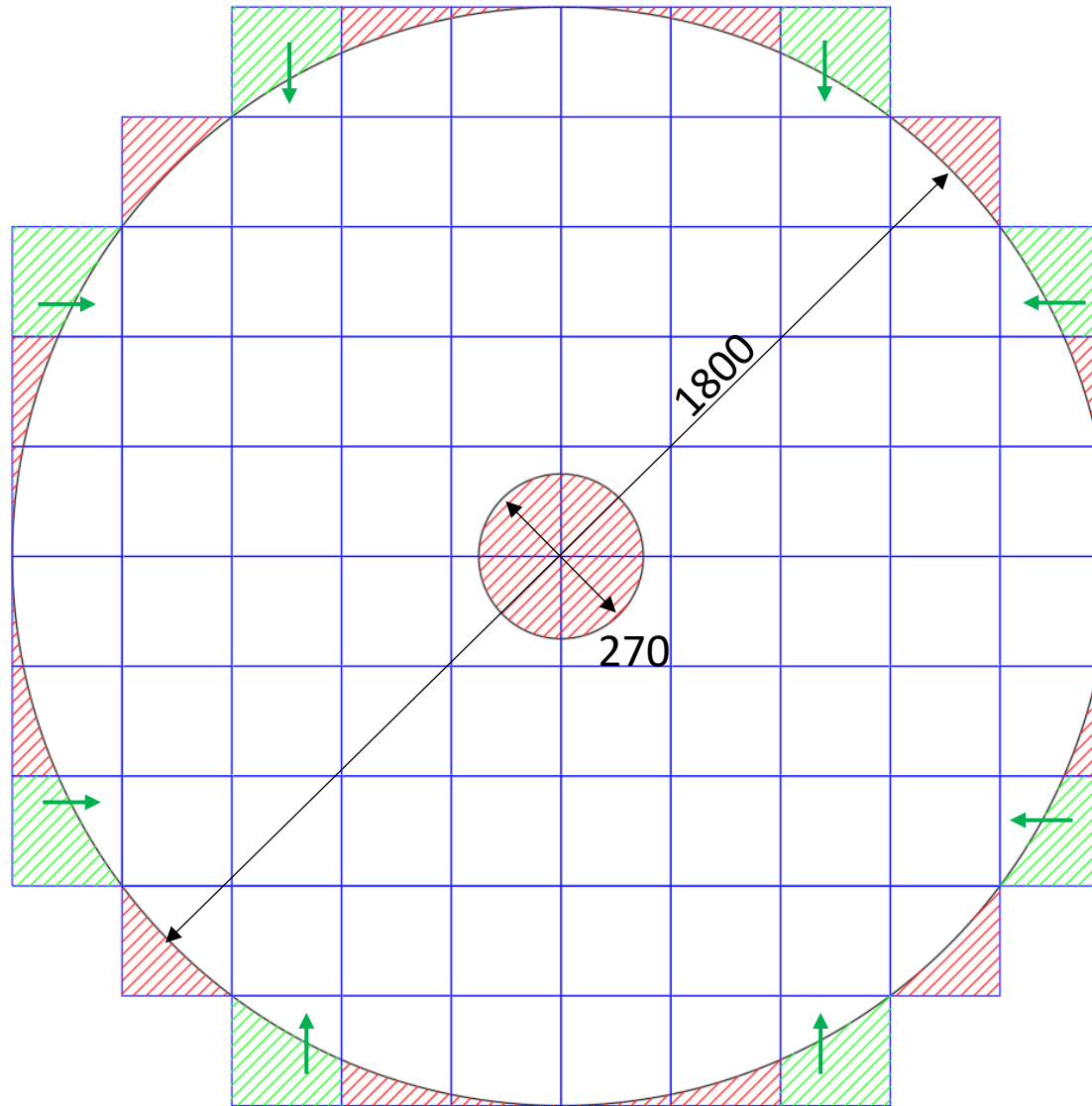


Active Area (A_a) = 21883,5 cm²

Dead Active Area (A_{da}) = $A_n - A_a$ = 2990,5 mm²
(12%)

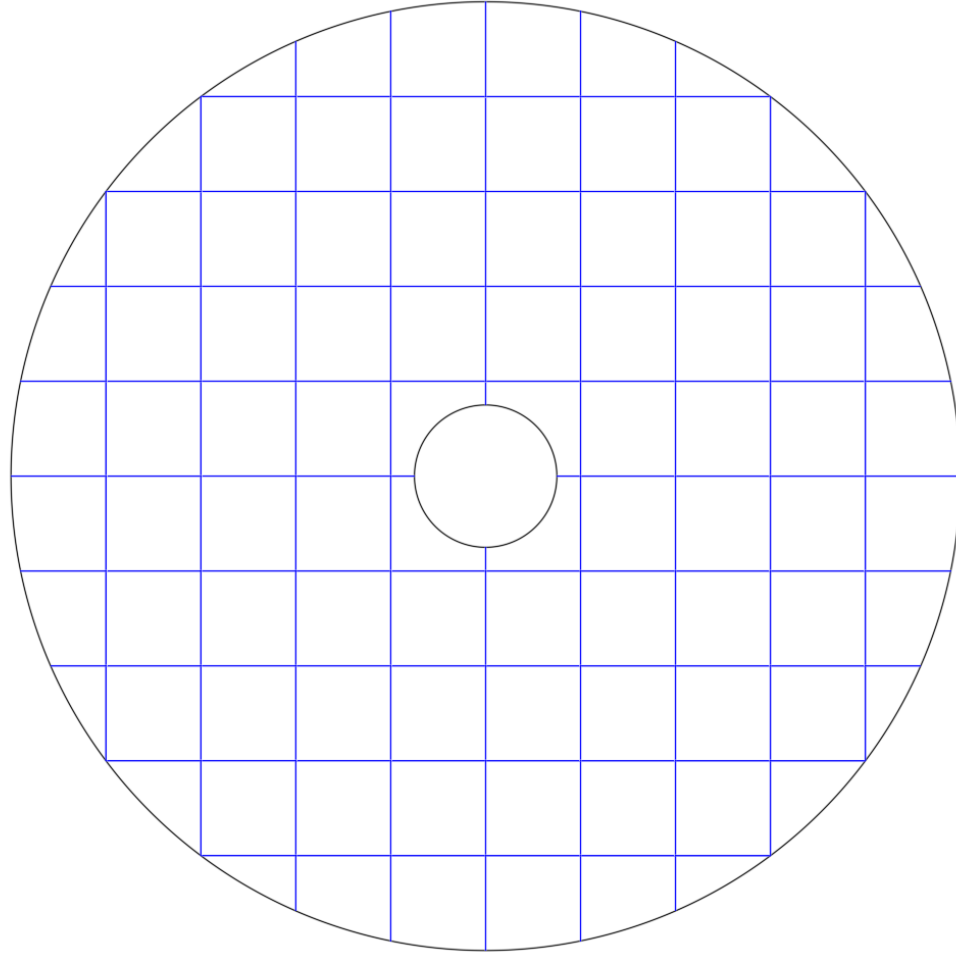
AEROGEL – Layout_B: nesting_B

84 tiles (one layer)
 $A_{\text{tile}} = 324 \text{ cm}^2$
 $A_{\text{tot,tiles}} = 27216 \text{ cm}^2$

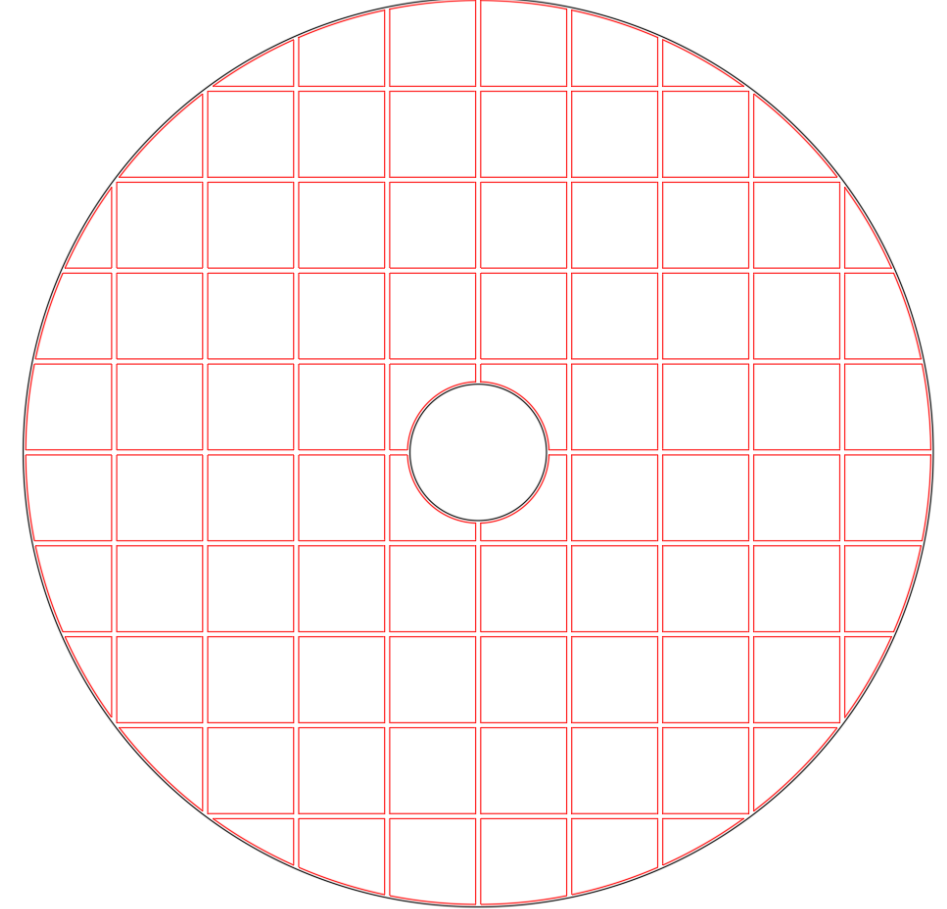


$A_{\text{twaste}} = 1868 \text{ cm}^2$
(6,8 %)

AEROGEL – Layout_B: Active Area and Dead Space



Nominal Active Area (A_n) = 24874 cm²



Active Area (A_a) = 21368 cm²

Dead Active Area (A_{da}) = $A_n - A_a$ = 3506 cm²
(14%)

AEROGEL – Layout_C: nesting_C

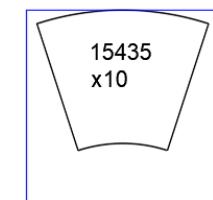
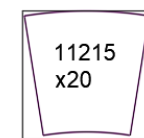
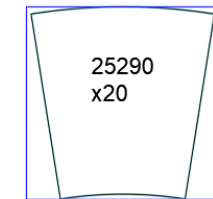
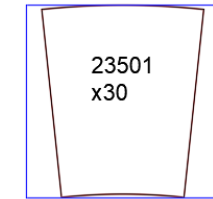
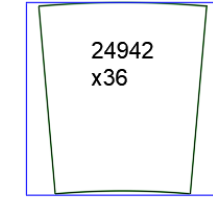
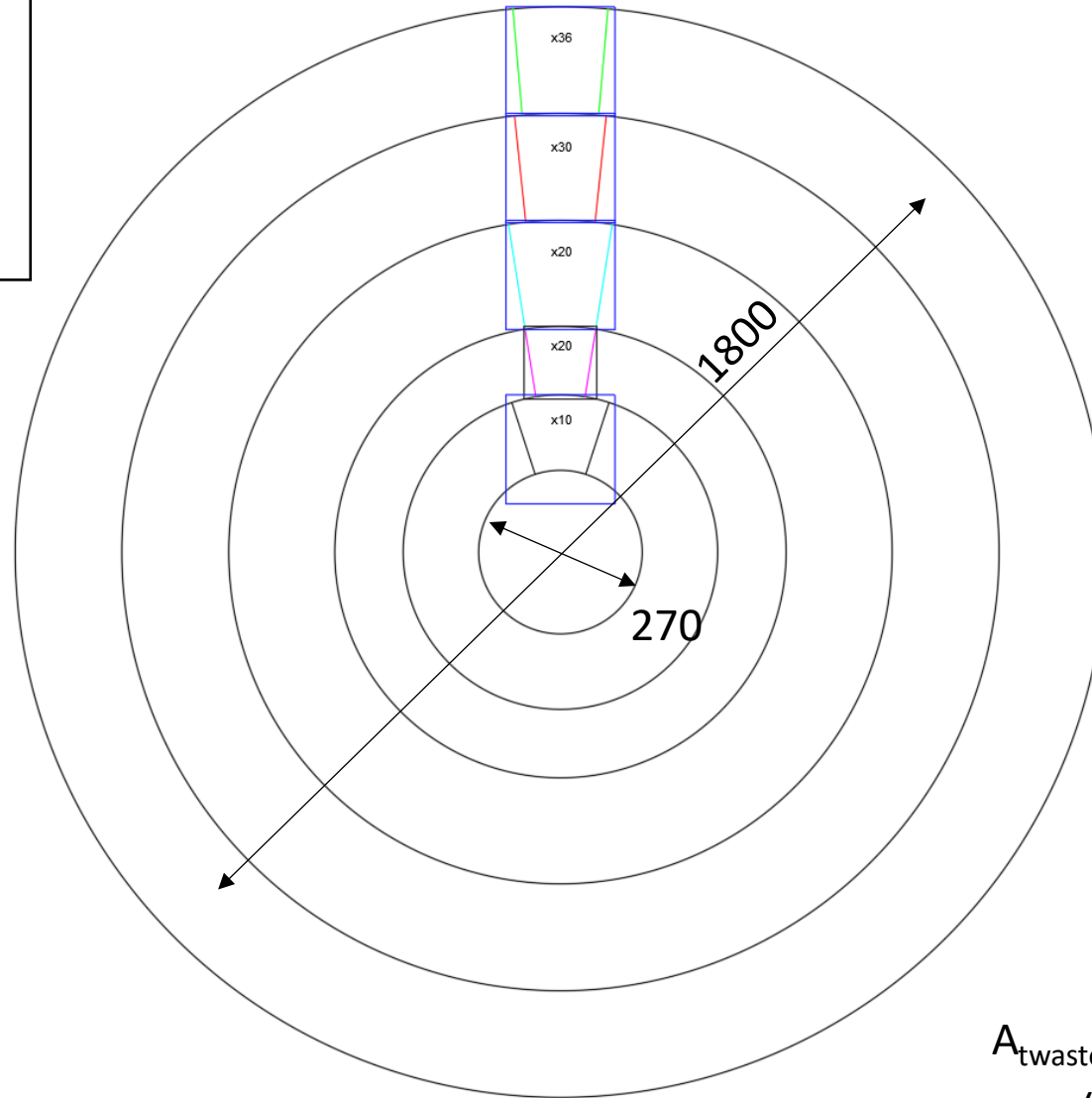
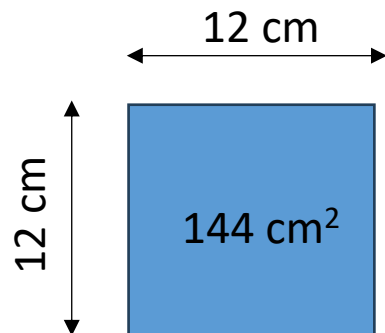
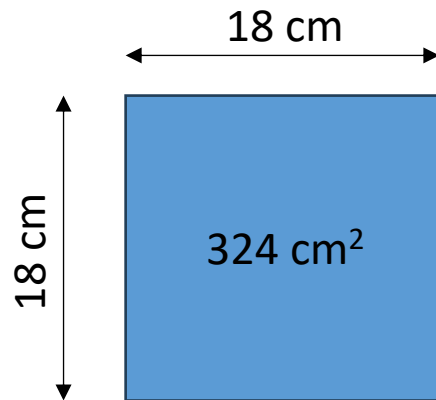
needs

116 tiles (one layer)

$$A_{\text{tile}} = 18\text{cm} \times 18\text{cm} = 324 \text{ cm}^2$$

$$A_{\text{tile}} = 12\text{cm} \times 12\text{cm} = 144 \text{ cm}^2$$

$$A_{\text{tot,tiles}} = 33984 \text{ cm}^2$$

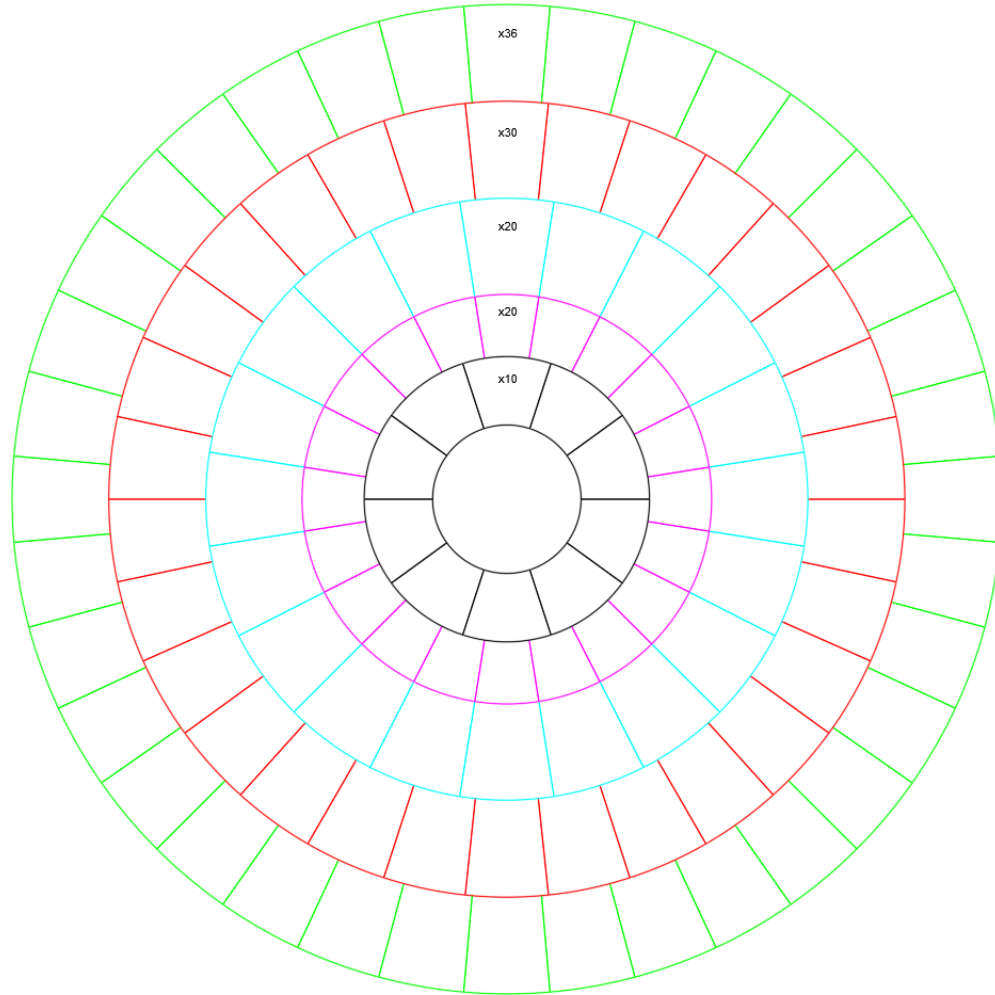


$$A_{\text{twaste}} = 9112 \text{ cm}^2$$

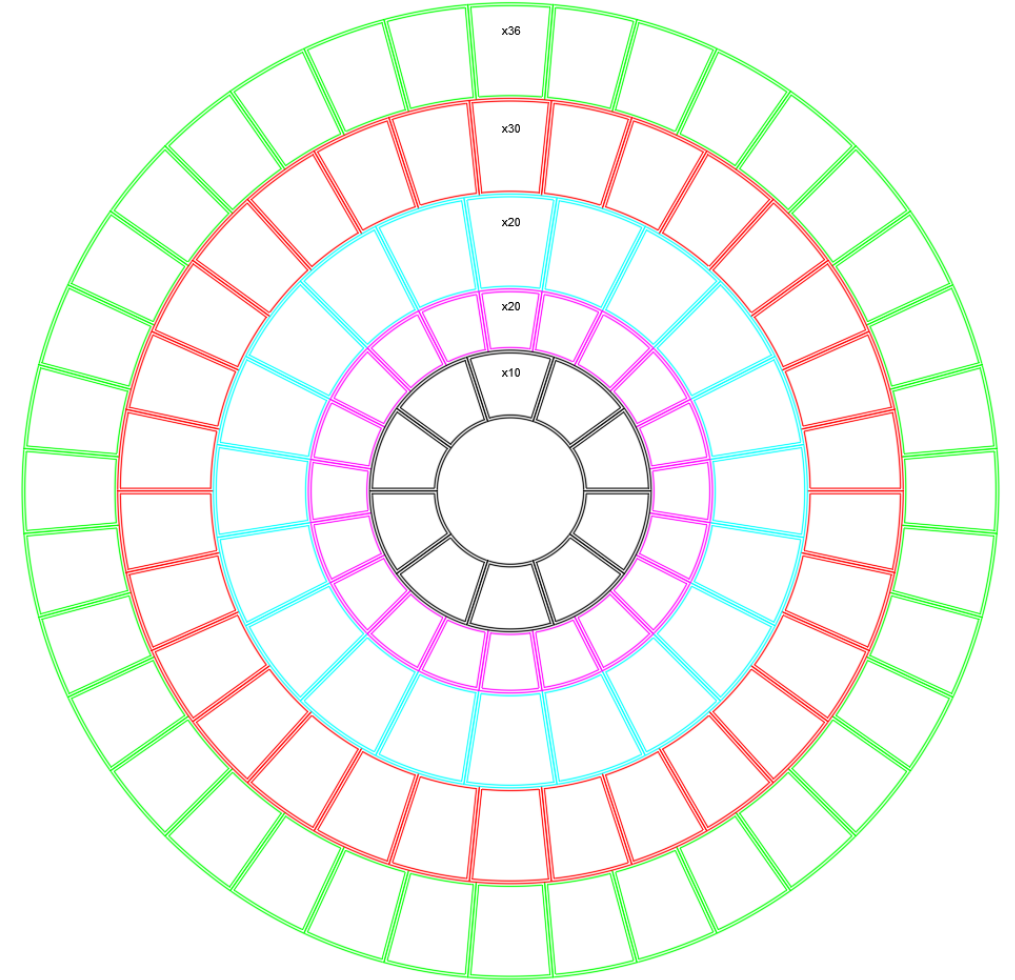
(26,8 %)

AEROGEL – Layout_C: Active Area and Dead Space

nil volentibus arduum



Nominal Active Area (A_n) = 24874 cm²



Active Area (A_a) = 21605 cm²
Dead Active Area (A_{da}) = $A_n - A_a$ = 3269 cm²
(13,1%)

PDU and boards: cooling and thermal power

Designed and
Engineered by
Roberto Preghenella
(INFN-BO)

dRICH PDU = 1200
Detector Box PDU = 242
dRICH Detector Boxes = 6

SiPM

$P_{\text{PDU}} = 5 \text{ W}$ (cooling power to be supplied to each PDU unit)

$T_{\text{SiPM}} = -40^\circ\text{C}$ (SiPM temperature)

$P_{\text{DT}} = 242 \times 5 \text{ W} = 1210 \text{ W}$ (cooling power to be supplied to each detector box)

$P_{\text{dRICH}} = 6 \times 1210 \text{ W} = 7260 \text{ W}$ (cooling power to be supplied to dRICH)

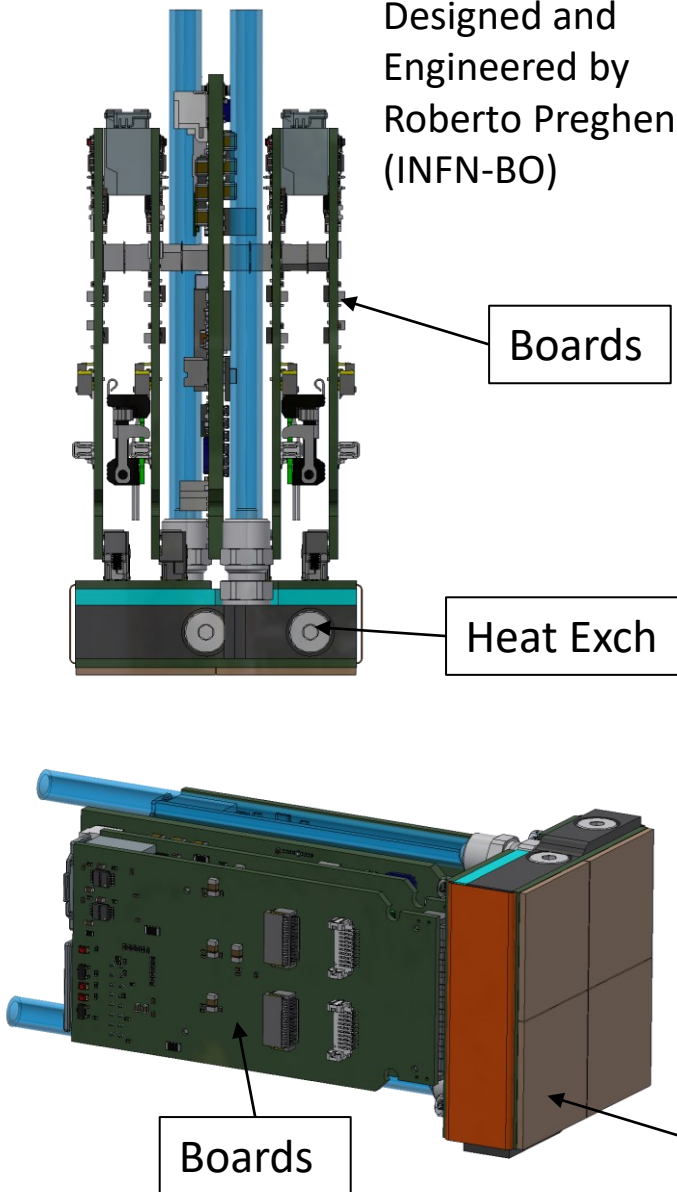
Electronic Boards

$P_{\text{boards}} = 11 \text{ W}$ (thermal power generated by each PDU unit)

$T_{\text{boards}} = 30^\circ\text{C}$ (maximum admissible boards temperature)

$P_{\text{BDT}} = 242 \times 11 \text{ W} = 2662 \text{ W}$ (thermal power generated by each detector box)

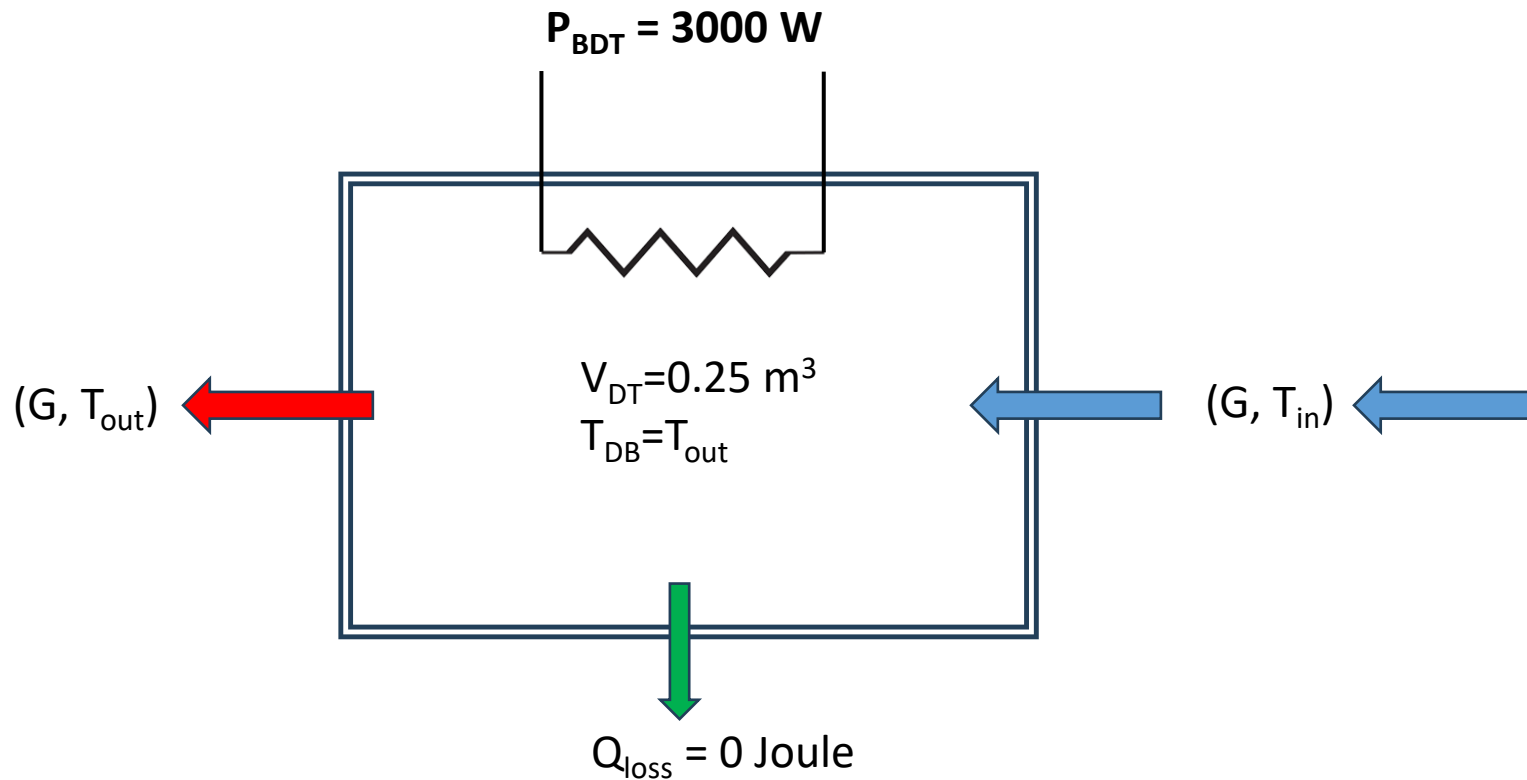
$P_{\text{dRICH}} = 6 \times 2662 \text{ W} = 15972 \text{ W}$ (thermal power generated by dRICH)



Detector Box: board's air cooling

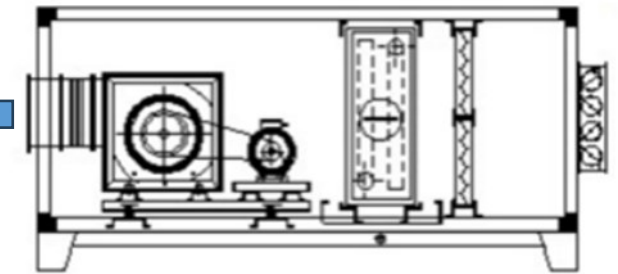
Hypothesis:

- Detector box perfectly insulated
- Thermal interaction with the SiPM cooling system not considered



Disadvantages:

- Risk of condensation
- High noise
- High vibrations
- Big size of ducts



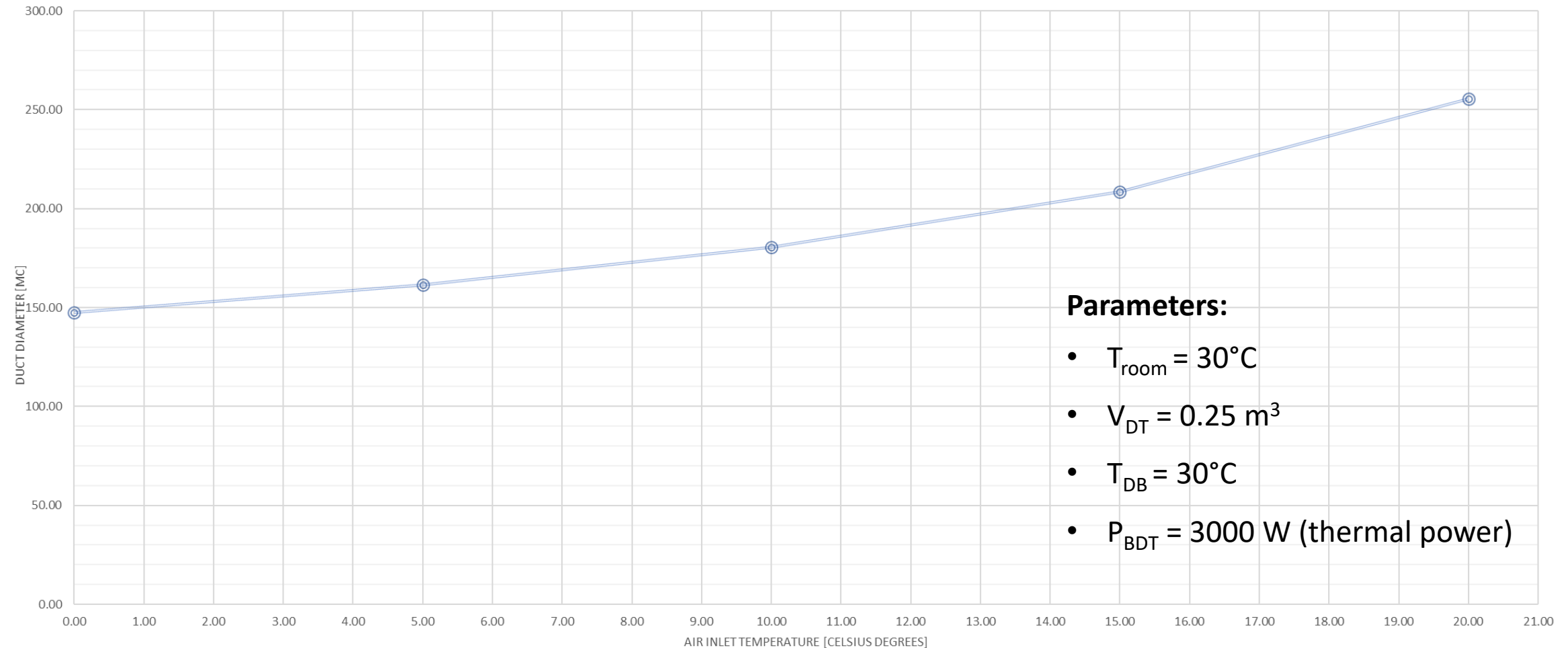
- External air intake
- Bag filter
- Cold coil
- Fan

Detector Box: board's air cooling

Disadvantages:

- Condensation
- Ducts size too big

Thermal Power = 3 kW
 $T_{\text{box}} = 30$ Celsius degrees
air speed = 5 m/s



Parameters:

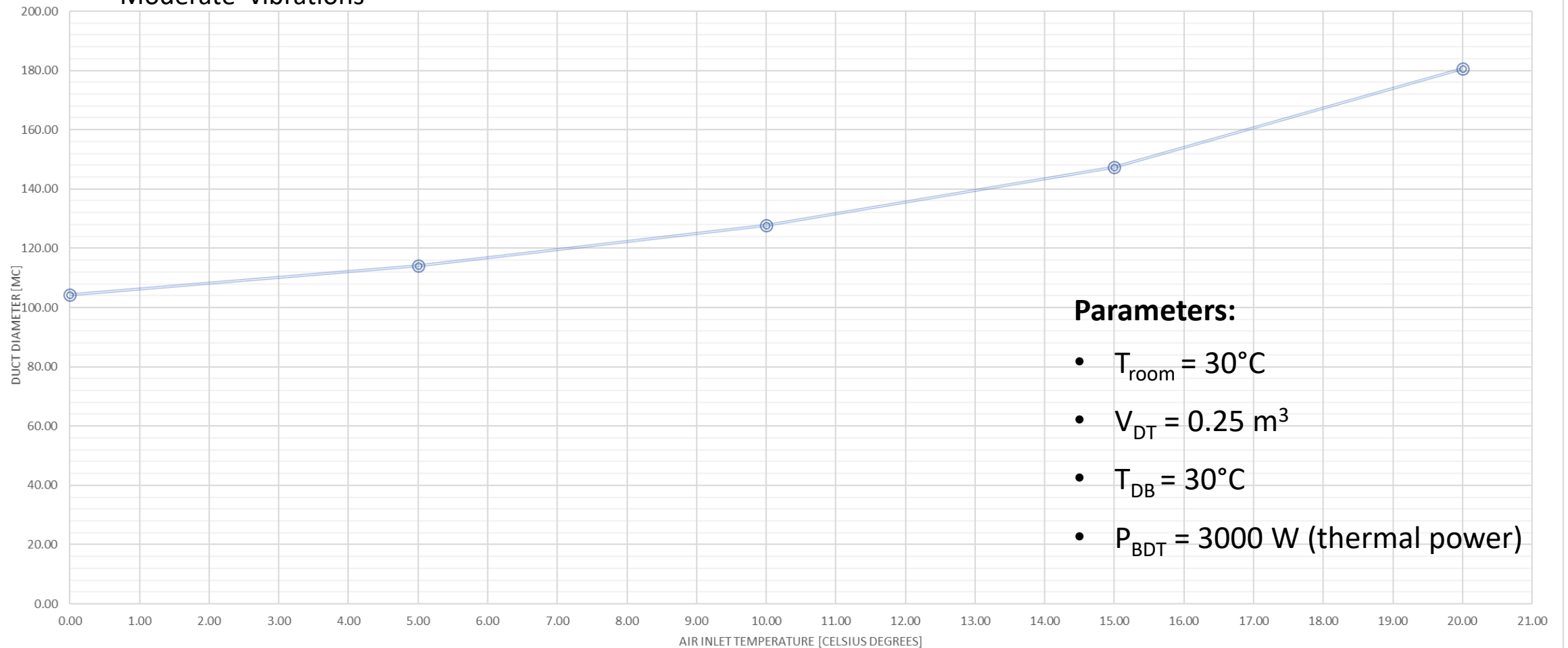
- $T_{\text{room}} = 30^{\circ}\text{C}$
- $V_{\text{DT}} = 0.25 \text{ m}^3$
- $T_{\text{DB}} = 30^{\circ}\text{C}$
- $P_{\text{BDT}} = 3000 \text{ W}$ (thermal power)

Detector Box: board's air cooling

Disadvantages:

- Condensation
- Moderate noise level
- Moderate vibrations

Thermal Power = 3 kW
 $T_{\text{box}} = 30$ Celsius degrees
air speed = 10 m/s



Parameters:

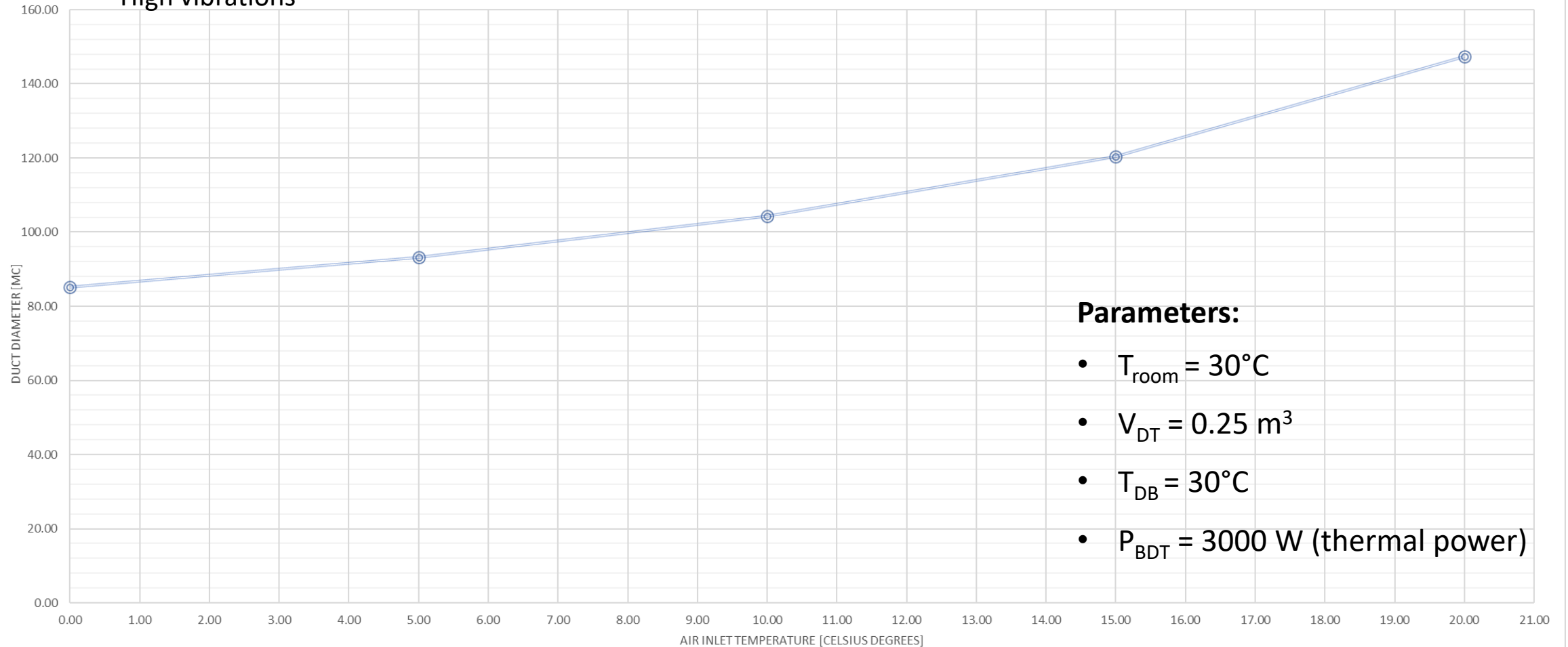
- $T_{\text{room}} = 30^{\circ}\text{C}$
- $V_{\text{DT}} = 0.25 \text{ m}^3$
- $T_{\text{DB}} = 30^{\circ}\text{C}$
- $P_{\text{BDT}} = 3000 \text{ W}$ (thermal power)

Detector Box: board's air cooling

Disadvantages:

- Condensation
- High noise
- High vibrations

Thermal Power = 3 kW
 $T_{\text{box}} = 30$ Celsius degrees
air speed = 15 m/s



Parameters:

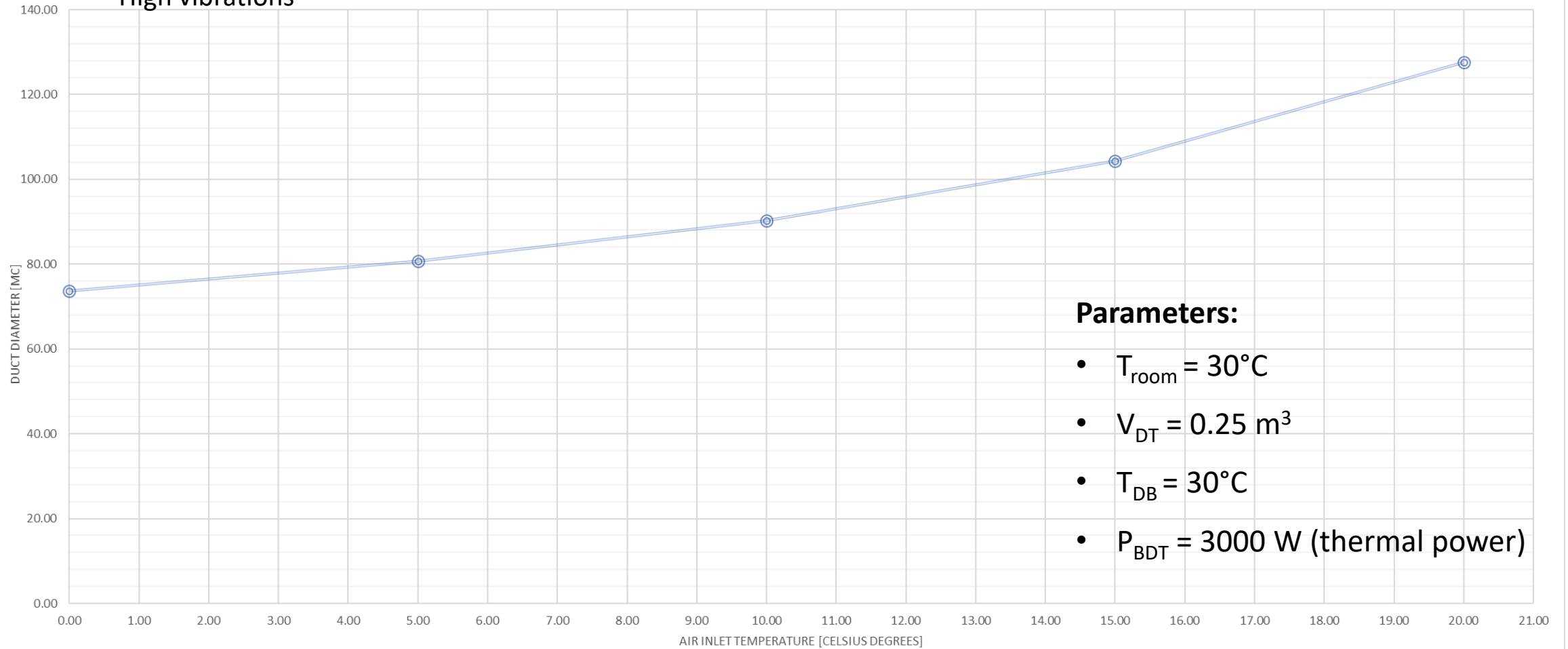
- $T_{\text{room}} = 30^{\circ}\text{C}$
- $V_{\text{DT}} = 0.25 \text{ m}^3$
- $T_{\text{DB}} = 30^{\circ}\text{C}$
- $P_{\text{BDT}} = 3000 \text{ W}$ (thermal power)

Detector Box: board's air cooling

Disadvantages:

- Condensation
- High noise
- High vibrations

Thermal Power = 3 kW
 $T_{\text{box}} = 30$ Celsius degrees
air speed = 20 m/s



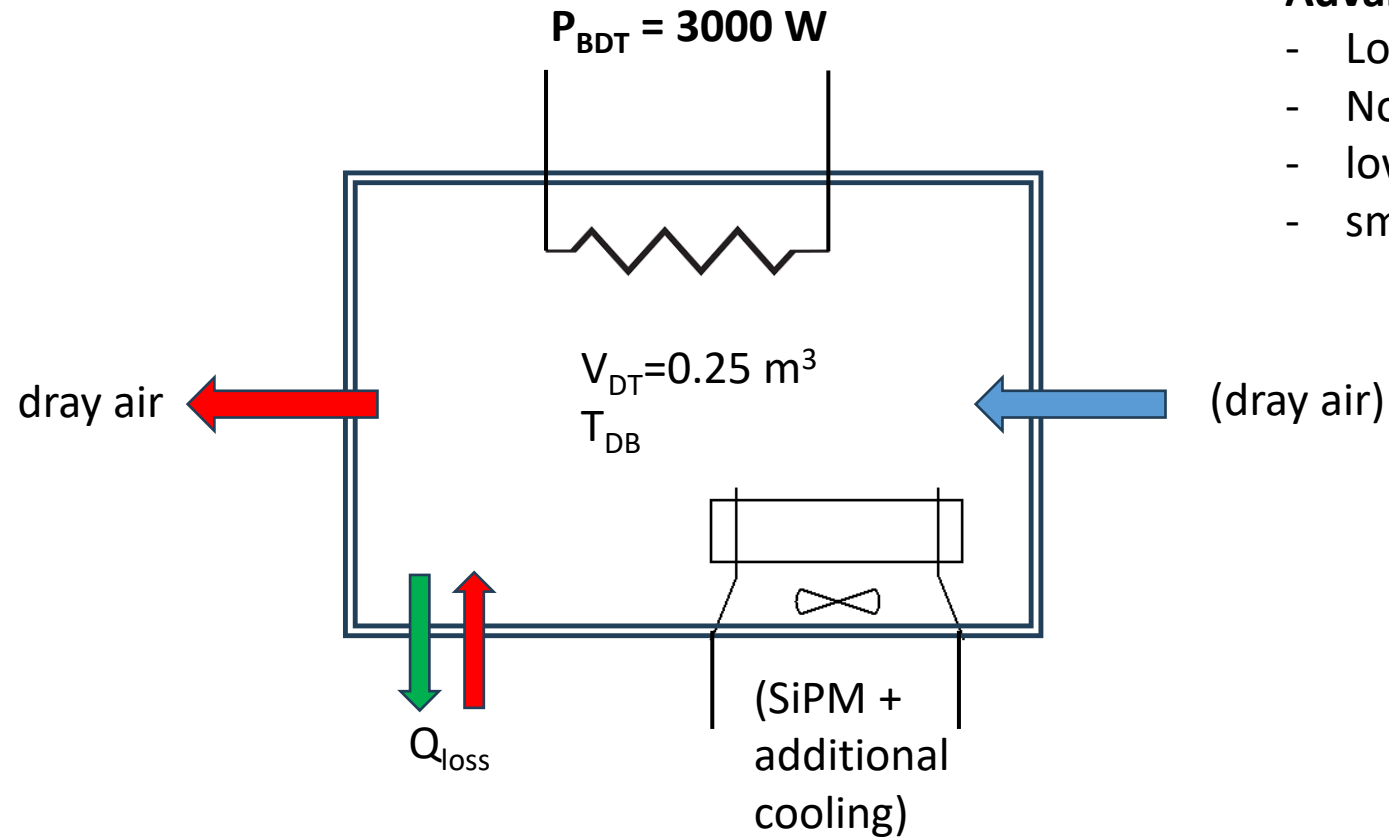
Parameters:

- $T_{\text{room}} = 30^{\circ}\text{C}$
- $V_{\text{DT}} = 0.25 \text{ m}^3$
- $T_{\text{DB}} = 30^{\circ}\text{C}$
- $P_{\text{BDT}} = 3000 \text{ W}$ (thermal power)

Detector Box: board's cooling

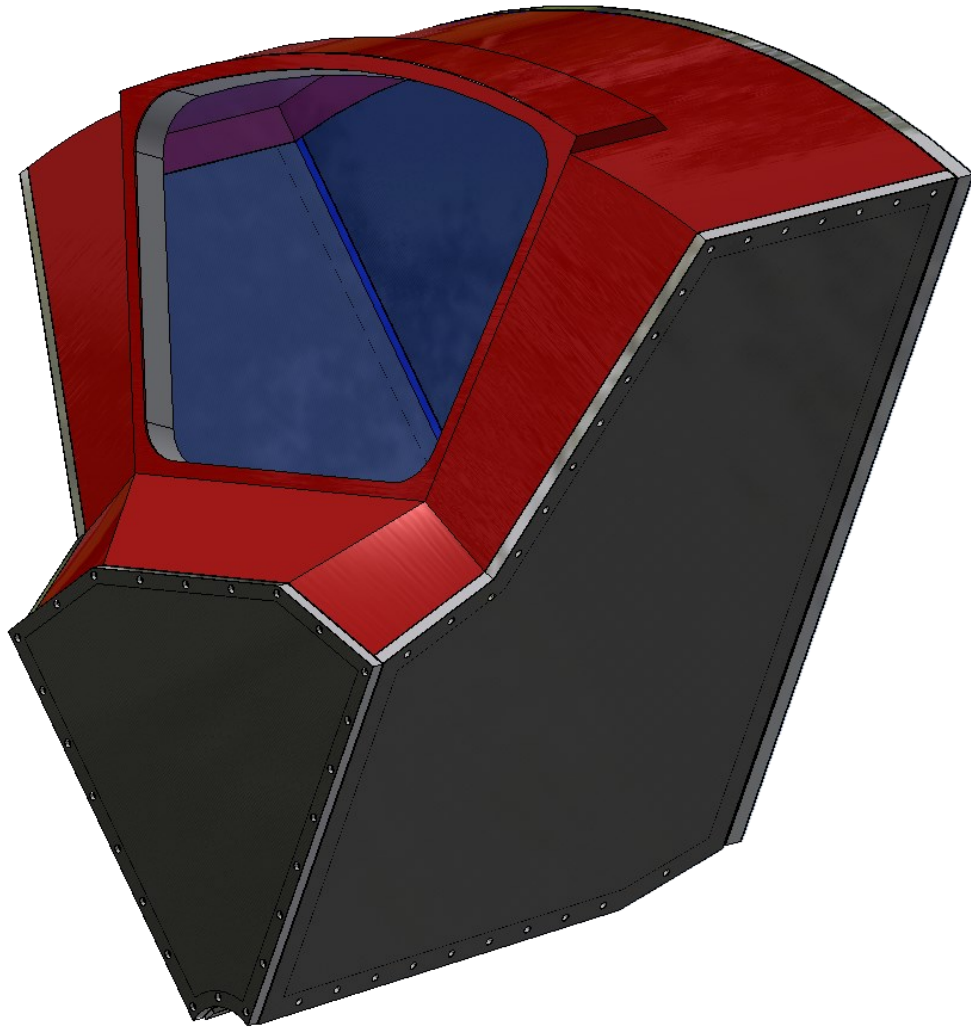
Hypothesis:

- Detector box no perfectly insulated
- Thermal interaction with the SiPM cooling system considered



Advantages:

- Low risk of condensation
- No noise
- low vibrations
- small size of pipes



Single Slice
(one-sixth)

Full-scale (1:1) prototype representing one-sixth of the complete dRICH detector.

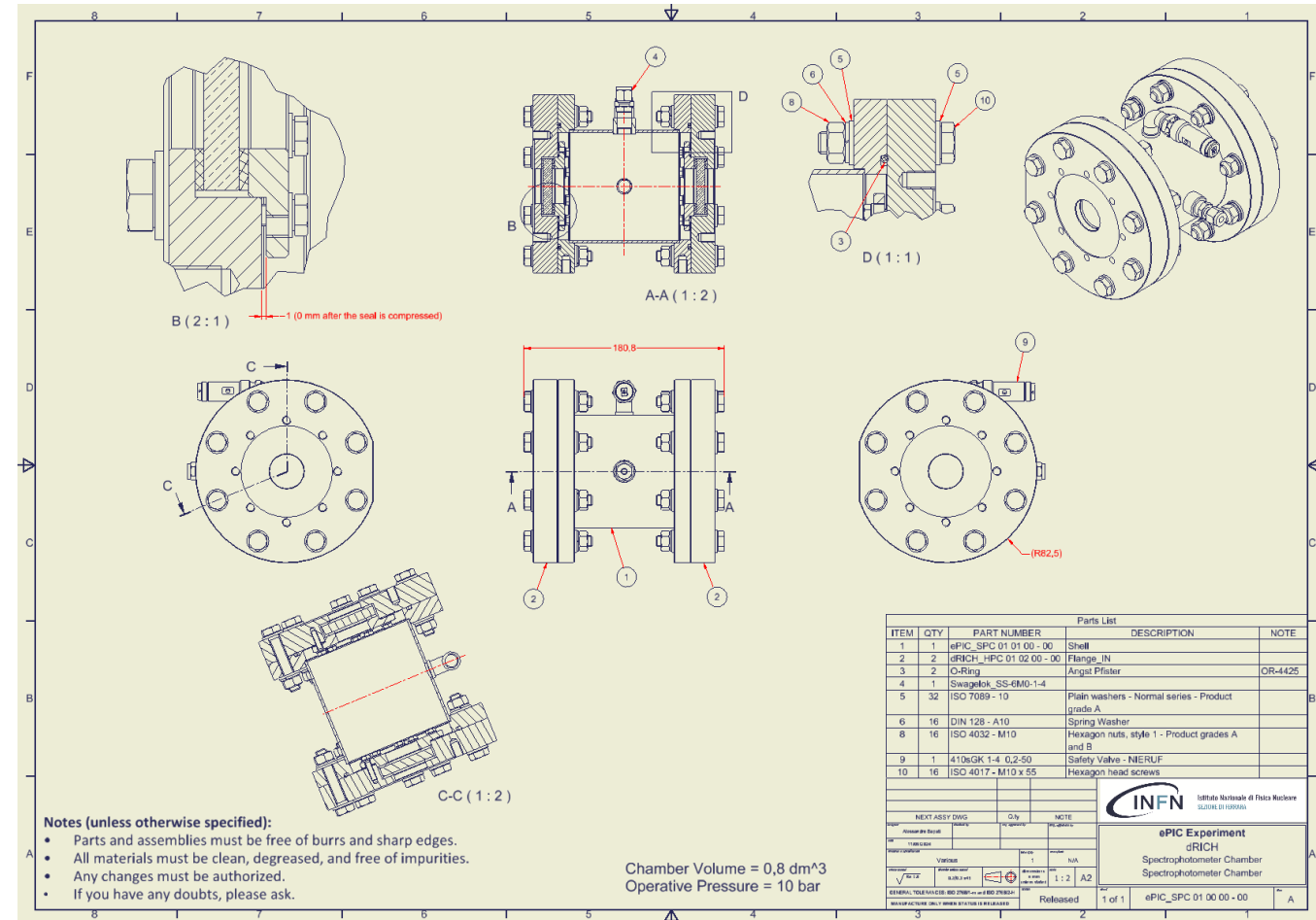
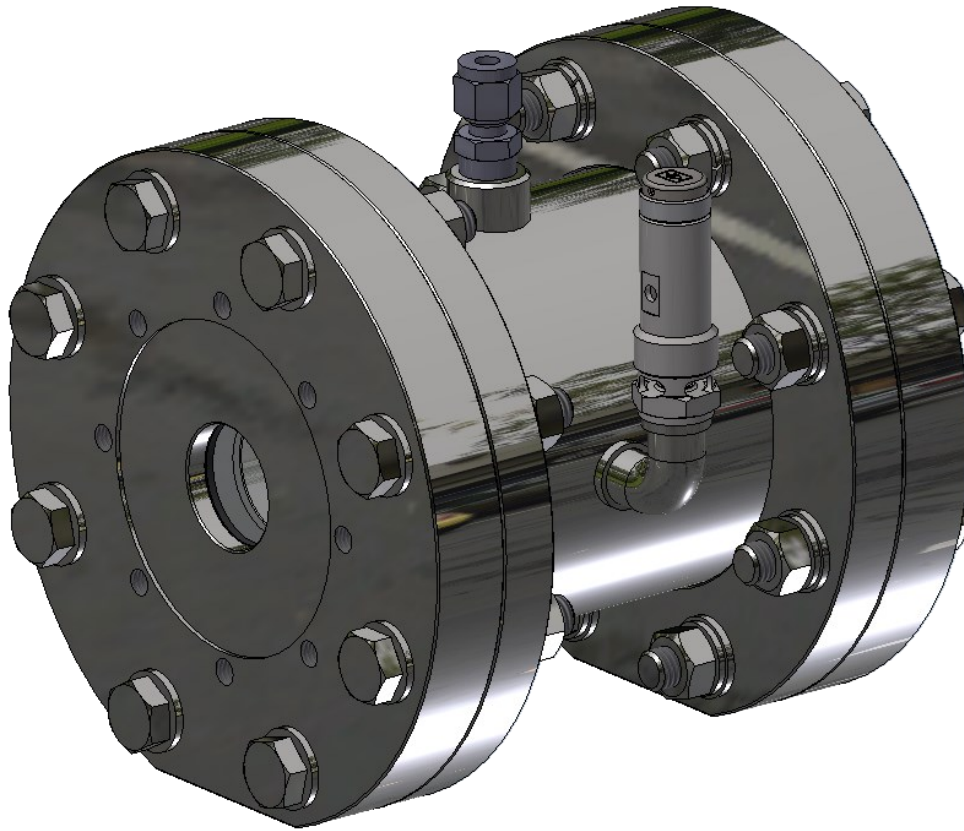
Status:

- The drawings are ready.
- The tender has been completed.
- Construction should start before the end of March 2025.

Spectrophotometer Chamber: target R&D

nil volentibus arduum

Design Pressure: 1,43 Mpa
Operative Pressure: 1 MPa

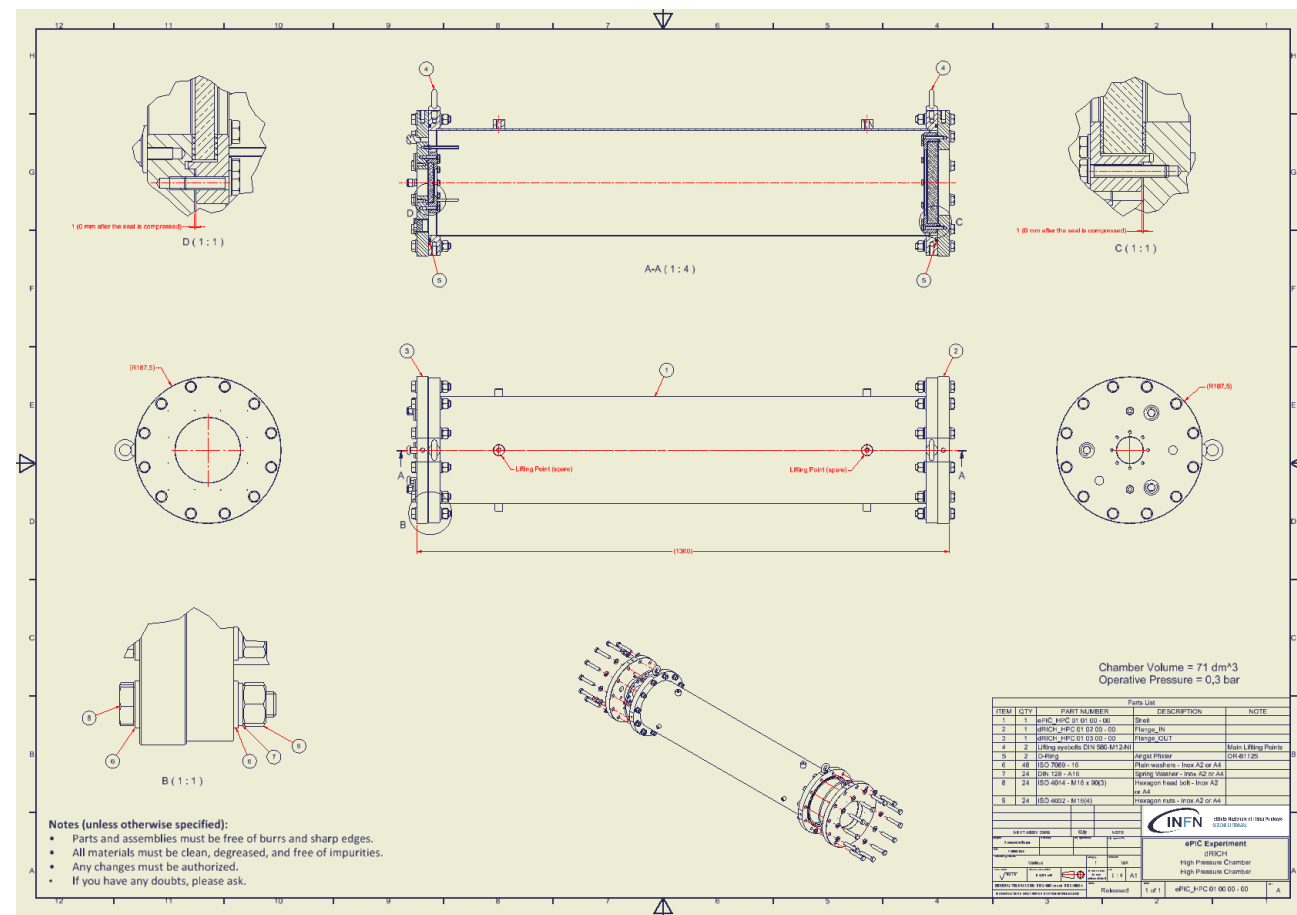


This chamber has been designed to study:

- Gas permeability of carbon fiber
- Gas transparency in the visible spectrum
- Gas-aerogel interactions

High Pressure Chamber: generic R&D

Design Pressure: 0,5 Mpa
Operative Pressure: 0,3 MPa

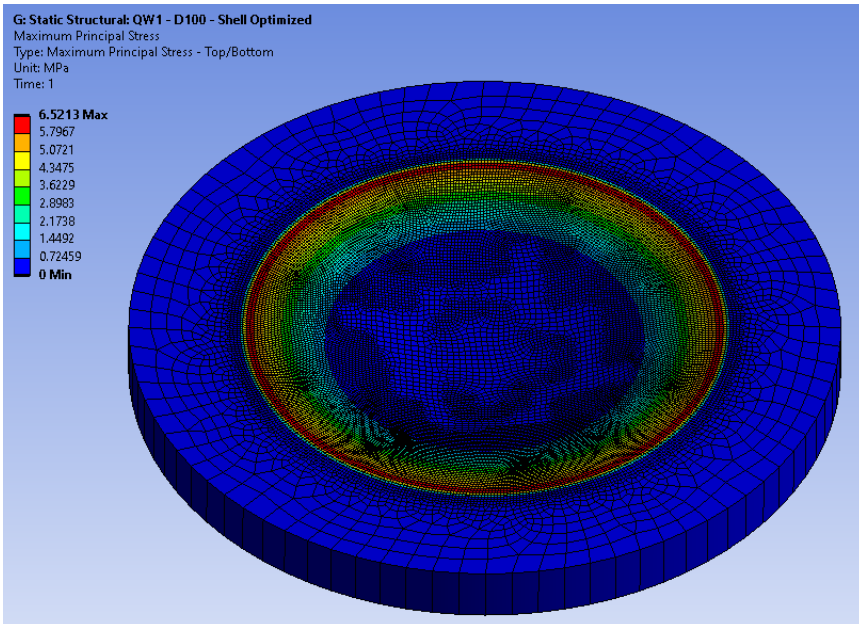


This chamber has been designed to study risk mitigation strategies for greenhouse gases.

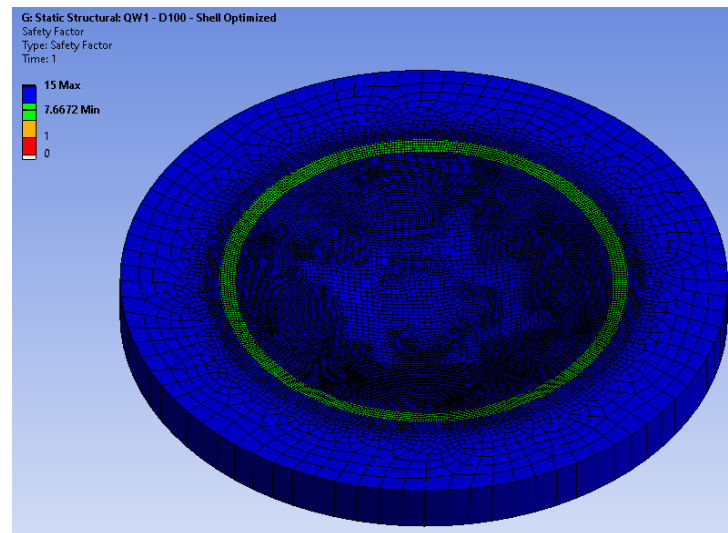
Quartz Window: stress&strain calculation - $t_1=8$ mm

- Thickness: $t_1=8$ mm;
- External diameter: $D_1=100$ mm
- Uniformly distributed (absolute) pressure over the entire surface ($d_1=68$ mm): 0,5 MPa
- Constrains: Fixed along the edge surface

Maximum Principal Stress (S1)



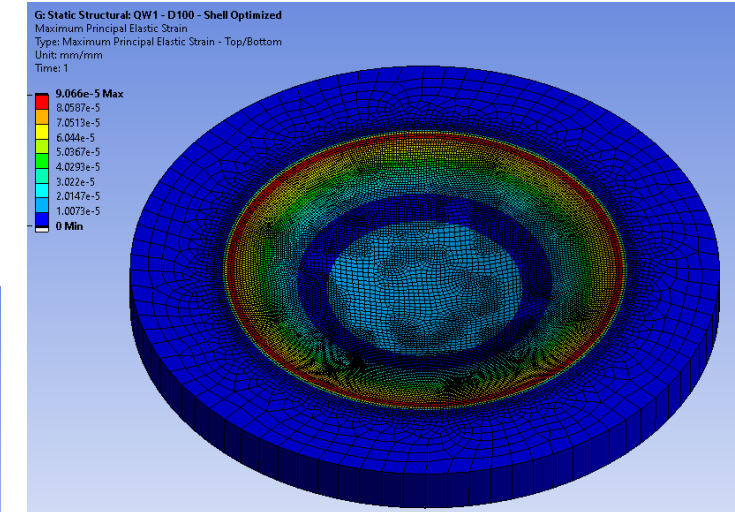
Safety Factor



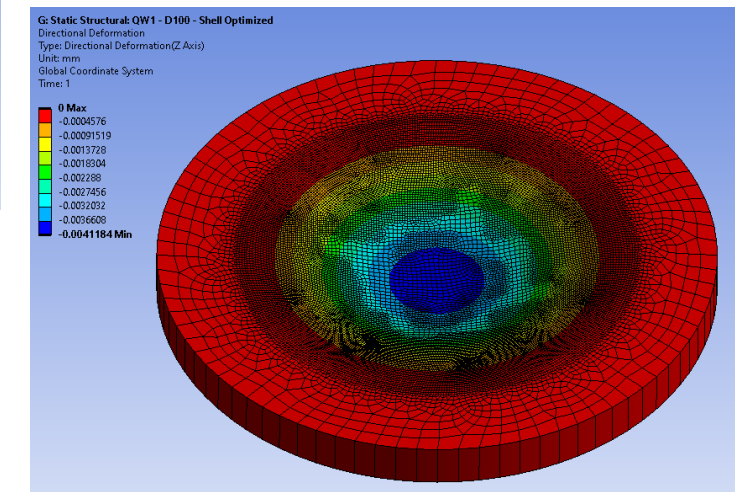
- Maximum Principal Stress $S1=6,5$ MPa;
- Maximum Principal Strain = 0,00009
- Maximum Deformation along Z Axis = -0,004 mm
- Minimum Calculated Safety Factor $SF_c = 7,6$

$SF_c > 7$ → checked

Maximum Principal Strain



Deformation along Z Axis



Conclusions and NEXT Steps

- **Consolidation of the dRICH Mechanical Design:** refining and finalizing the mechanical design of the dRICH
- **Integration study of dRICH into the ePIC Apparatus:** fixing system, service integration.....
- **Study of the Extraction and Insertion System (Moving System):** design and optimization of the moving system used for the extraction and insertion of the dRICH detector within the ePIC apparatus
- **Structural Study of dRICH:** A detailed structural analysis of the dRICH detector will be conducted using Finite Element Method (FEM) simulations.

